

Quelques exemples de dynamique non-linéaire dans la mécanique du contact/frottement

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CNRS UMR 7633, Evry, France*



Manifestation du GDR DYNOLIN
Dynamique **NO**n **LIN**éaire
ENSTA ParisTech, Palaiseau, France
11 octobre 2016

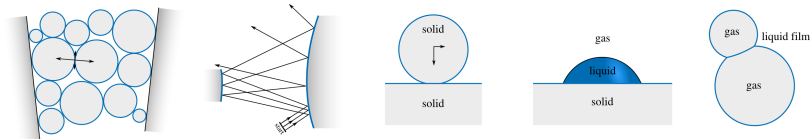
• Outline

- Introduction to contact mechanics
 - Industrial and natural problems
 - Complex interface physics
 - Several important examples
- Elastodynamic frictional sliding on bi-material interface
 - *Vibration of asymmetric materials with internal contacts*

Introduction

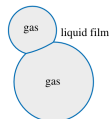
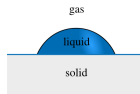
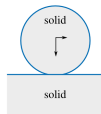
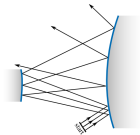
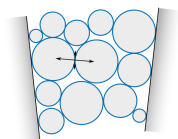
• Introduction

- Dynamical systems are often defined over a single independent variable: scalar time t
- Multi-/infinite-dimensional extension: loading conditions (tractions, displacements): $\underline{p}(x, t)$ for $x \in \Gamma_f(t), t \in \mathbb{R}$
- Examples of dynamical systems with contacts
 - Granular matter
 - Dynamical billiards
 - Mechanical contact between solids (quasi-static/dynamic)
 - Liquid/solid contact (e.g., *liquid drop on a substrate*)
 - Liquid/liquid contact (e.g., *merger of liquid drops*)



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• Industrial and natural contact problems

■ Tyre/road contact



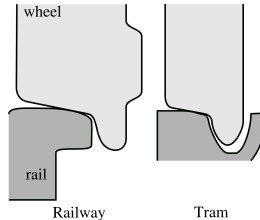
www.motortrend.com

• Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact



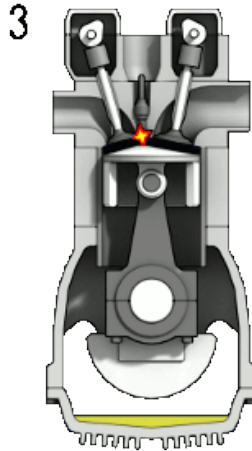
Railway wheel www.railway-wheel-axle.com



Railway Tram
Wheel flange
adapted from Wikipedia

• Industrial and natural contact problems

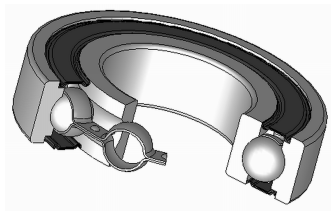
- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder



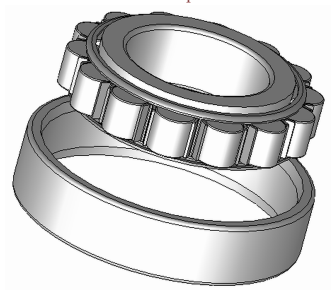
Four-stroke cycle in cylinder
from [Wikipedia](#)

• Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder
- Bearings



A sealed deep groove ball bearing
from [Wikipedia](#)



A cylindrical roller bearing
from [Wikipedia](#)

• Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder
- Bearings
- Gears



Change gear



Helical gear



Bevel gear

www.linngear.com

• Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder
- Bearings
- Gears



Interior of Rolex watches

www.rolex.com



Ship reduction gearbox 14 MW
(e.g. Renault Mégane 1.4 $\approx$ 60 KW)

www.renk.eu

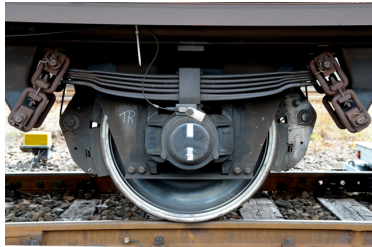
• Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder
- Bearings
- Gears
- Brake systems



Reinforced carbon brake disc on a Ferrari F430

[Wikipedia](#)



New LL brake blocks aimed to reduce noise from rail sector [photo: UIC/EuropeTrain](#)

• Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder
- Bearings
- Gears
- Brake systems
- Assembled pieces



Modern steam turbine [Wikipedia](#)



Fuselage of modern aircrafts contains $\approx 100\,000$ rivets

www.news.cn

● Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder
- Bearings
- Gears
- Brake systems
- Assembled pieces
- Impact and crash



Bird impact traces on aircraft's nose



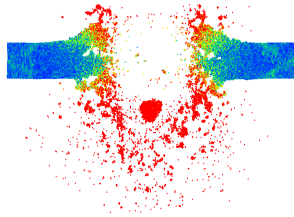
Mercedes crash test
Insurance Institute for Highway Safety

• Industrial and natural contact problems

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- Assembled pieces
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- Penetration and perforation



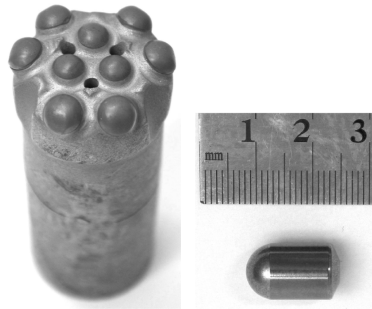
Sherman Firefly armor piercing shell on Tiger tank armor, Bovington Tank Museum
Andy's photo www.flickr.com



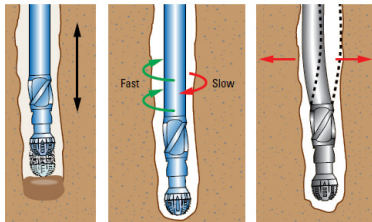
*Hyper velocity perforation
Molecular Dynamics simulation*

• Industrial and natural contact problems

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- Brake systems
- Assembled pieces
- Impact and crash
- Penetration and perforation
- Drilling and cutting



Drill crown and a single drill-bit button WC-Co
Bit bounce **Stick/slip** **Bending**



Axial

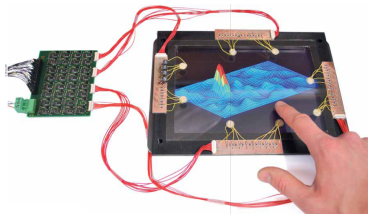
Torsional

Lateral

Drilling column vibrations

• Industrial and natural contact problems

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- Drilling and cutting
- Haptic perception



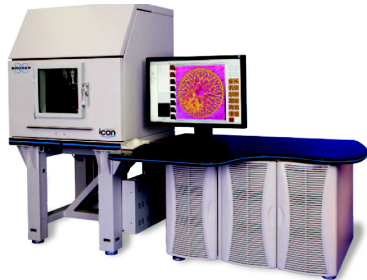
Sensory interacting system
V. Hayward, ISIR UPMC
CNRS International Magazine 34 (2014)



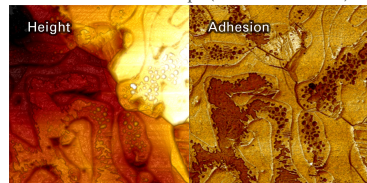
Braille page www.todayifoundout.com

● Industrial and natural contact problems

- Tyre/road contact
- Wheel/rail contact
- Piston/cylinder
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- Gears
- Brake systems
- Assembled pieces
- Impact and crash
- Penetration and perforation
- Drilling and cutting
- Haptic perception
- Atomic force microscopy



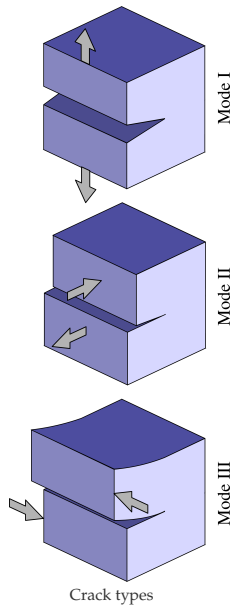
Atomic force microscope (www.bruker.com)



Height and adhesion measurements of Sn-Pb alloy surface (AFM)
www.bruker.com

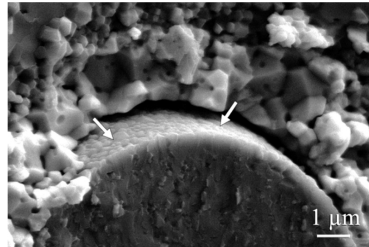
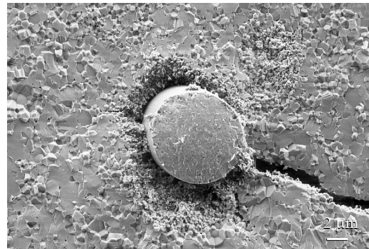
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Fiber-matrix interface

[1] D. Blaese et al. ZrO_2 fiber-matrix interfaces in alumina fiber-reinforced model composites, J Eur Ceramic Soc 35 (2015)

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- Haptic perception
- Atomic force microscopy
- Interface cracks
- Electrical contact



Siemens Switch www.siemens.com



Rouen's tram brush
[Wikipedia](#)

• Industrial and natural contact problems

- Tyre/road contact
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- Haptic perception
- Atomic force microscopy
- Interface cracks
- Electrical contact
- Music instruments & sound



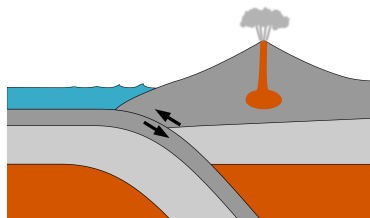
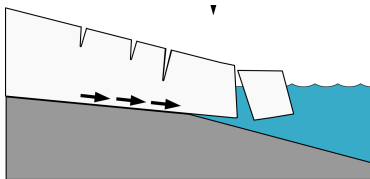
Violin and bow
www.walmart.com



Grasshopper's leg
by Nico Angleys on www.flickr.com

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- Electrical contact
- Music instruments & sound
- Geophysical contacts

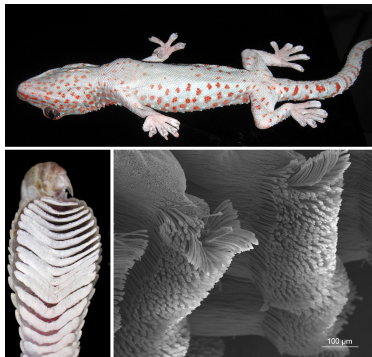


0.1-100 km

Geophysical scale slip
- basal glacial slip on the bedrock
- rock-rock slip in faults

● Industrial and natural contact problems

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- Geophysical contacts



Gecko adhesive toes

- Adhesion
- etc ...

• Frictionless or normal contact conditions

■ No penetration

Always non-negative gap

$$g \geq 0$$

■ No adhesion

Always non-positive contact pressure

$$\sigma_n^* \leq 0$$

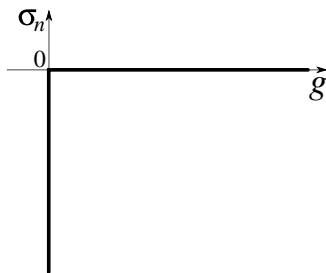
■ Complementary condition

Either zero gap and non-zero pressure, or non-zero gap and zero pressure

$$g \sigma_n = 0$$

■ No shear transfer (automatically)

$$\underline{\sigma}_t^{**} = 0$$



Scheme explaining normal contact conditions

$$\sigma_n^* = (\underline{\underline{\sigma}} \cdot \underline{n}) \cdot \underline{n} = \underline{\underline{\sigma}} : (\underline{n} \otimes \underline{n})$$

$$\underline{\sigma}_t^{**} = \underline{\underline{\sigma}} \cdot \underline{n} - \sigma_n \underline{n} = \underline{n} \cdot \underline{\underline{\sigma}} \cdot (\underline{I} - \underline{n} \otimes \underline{n})$$

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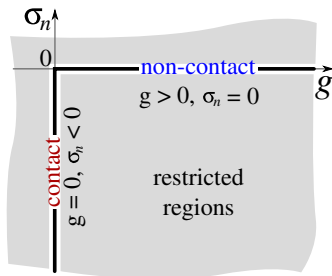
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Improved scheme explaining normal contact conditions

• Frictionless or normal contact conditions

In mechanics:

Normal contact conditions

$\equiv$

Frictionless contact conditions

$\equiv$

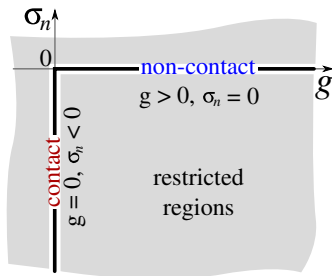
Hertz^[1]-Signorini^[2] conditions

$\equiv$

Hertz^[1]-Signorini^[2]-Moreau^[3] conditions

also known in **optimization theory** as

Karush^[4]-Kuhn^[5]-Tucker^[6] conditions



Improved scheme explaining
normal contact conditions

$$g \geq 0, \quad \sigma_n \leq 0, \quad g\sigma_n = 0$$

¹Heinrich Rudolf Hertz (1857–1894) a German physicist who first formulated and solved the frictionless contact problem between elastic ellipsoidal bodies.

²Antonio Signorini (1888–1963) an Italian mathematical physicist who gave a general and rigorous mathematical formulation of contact constraints.

³Jean Jacques Moreau (1923–2014) a French mathematician who formulated a non-convex optimization problem based on these conditions and introduced pseudo-potentials in contact mechanics.

⁴William Karush (1917–1997), ⁵Harold William Kuhn (1925) American mathematicians,

⁶Albert William Tucker (1905–1995) a Canadian mathematician.

• Amontons-Coulomb's friction

- **No contact** $g > 0, \sigma_n = 0$

- **Stick** $|\underline{v}_t| = 0$

Inside slip surface/Coulomb's cone

$$F = |\underline{\sigma}_t| - f|\sigma_n| < 0$$

- **Slip** $|\underline{v}_t| > 0$

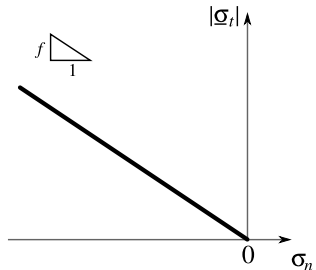
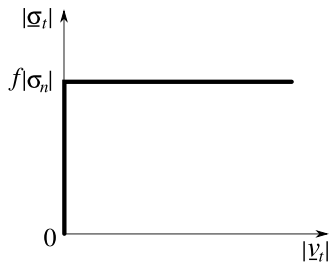
On slip surface/Coulomb's cone

$$F = |\underline{\sigma}_t| - f|\sigma_n| = 0$$

- **Complementary condition**

Either zero velocity and negative slip criterion, or non-zero velocity and zero slip criterion

$$|\underline{v}_t| \left(|\underline{\sigma}_t| - f|\sigma_n| \right) = 0$$



Scheme explaining frictional contact conditions

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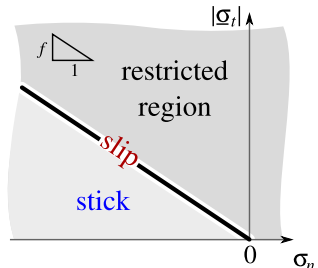
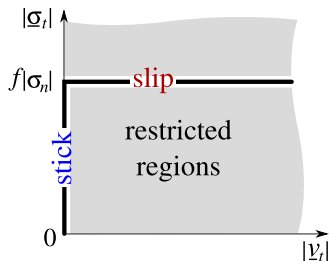
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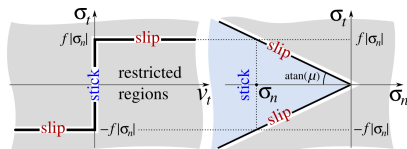
On slip surface/Coulomb's cone

$$F = |\underline{\sigma}_t| - f|\sigma_n| = 0$$

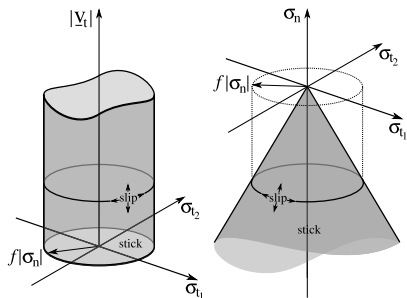
- **Complementary condition**

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Scheme of 2D frictional contact

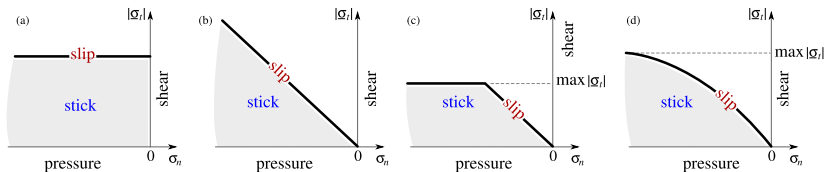


Scheme of 3D frictional contact

$$|\underline{v}_t| \geq 0, \quad |\underline{\sigma}_t| - f|\sigma_n| \leq 0, \quad |\underline{v}_t| \left(|\underline{\sigma}_t| - f|\sigma_n| \right) = 0$$

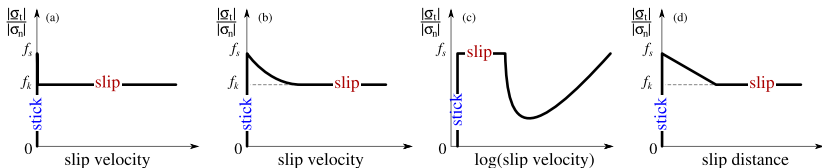
• Other friction laws I

• Static criteria



(a) Tresca (b) Amontons-Coulomb (c) Coulomb-Orowan (d) Shaw

• Kinetic criteria



(a,b) velocity weakening (c) velocity weakening-strengthening
(d) Linear slip weakening

- f_s static and f_k kinetic coefficients of friction.

• History dependent friction

• Rate and state friction law

- Rate $v_t = |\underline{v}_t|$ – relative slip velocity
- State $\theta \approx$ internal time
- Dieterich–Ruina–Perrin (1979, 83, 95)

Frictional resistance

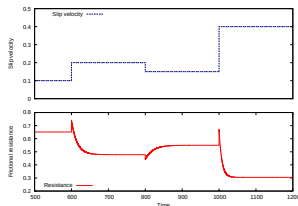
$$\sigma_t^c = |\sigma_n| [f_s + b\theta + a \ln(v_t/v_0)]$$

Evolution of the state variable

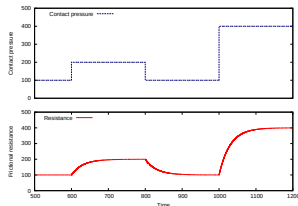
$$\dot{\theta} = -\frac{v_t}{L} \left[\theta + \ln\left(\frac{v_t}{v_0}\right) \right]$$

• Prakash-Clifton friction law (1992,2000)

- Viscous type evolution of frictional resistance σ_t
- $\dot{\sigma}_t = -\frac{v_t}{L} (\sigma_t + f\sigma_n)$

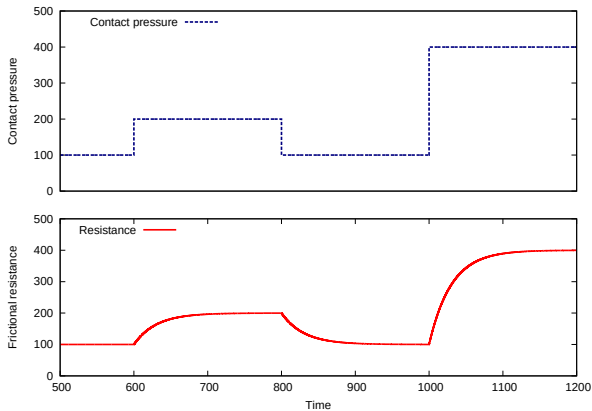


Rate and state friction law



Prakash-Clifton regularization

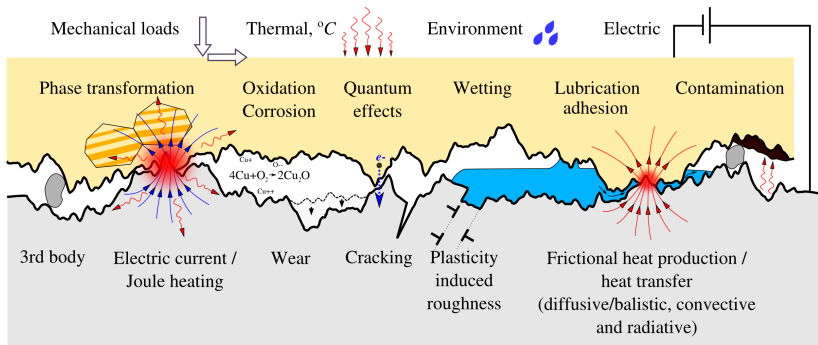
- History dependent friction



• Contact interface physics

Interface constitutive laws are determined by:

- Two distinct materials
- Two distinct roughness
- Thermal/electro-magnetic/mechanical loads
- Fluid/3rd body/tribofilm in the interface



V. Yastrebov "Contact Mechanics and Elements of Tribology", DMS master course.

- Example 1: contact between rough surfaces

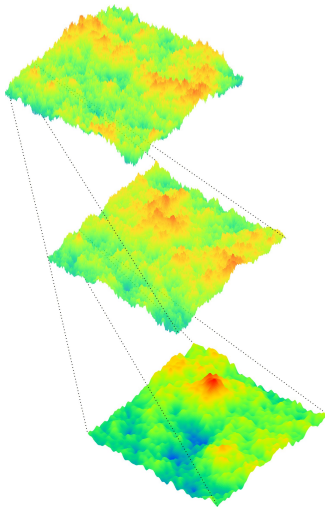


Fig. Example of a rough surface for $H = 0.3$

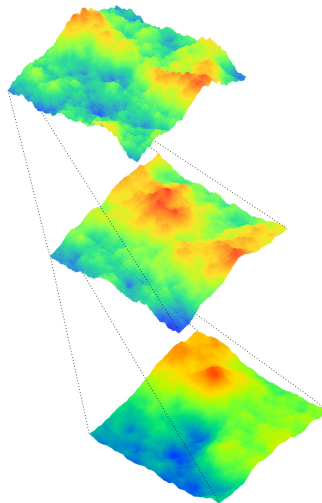


Fig. Example of a rough surface for $H = 0.8$

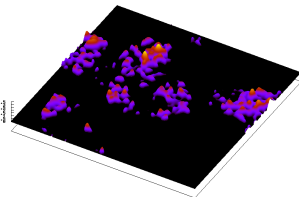
Recall: the Hurst exponent H and the fractal dimension D for 2-manifolds are interconnected via $D = 3 - H$

[1] Yastrebov, Anciaux, Molinari, Phys Rev E 86 (2012)

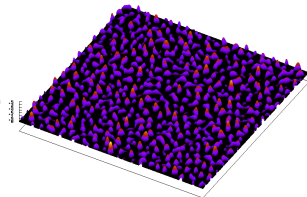
• Example 1: contact between rough surfaces

- Normal frictionless and non-adhesive contact between rough surfaces
- Dynamics of contact clusters

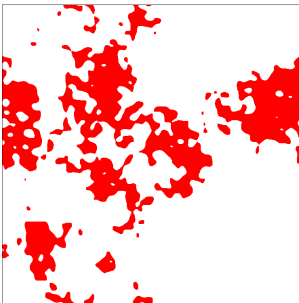
$k_t=1, k_s=32, H=0.8$
Contact pressure, $p(x,y)$



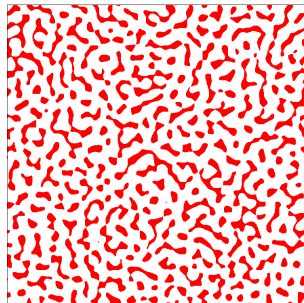
$k_t=16, k_s=32, H=0.8$
Contact pressure, $p(x,y)$



$k_t=1, k_s=32, H=0.8$
Contact area, $a(x,y)$



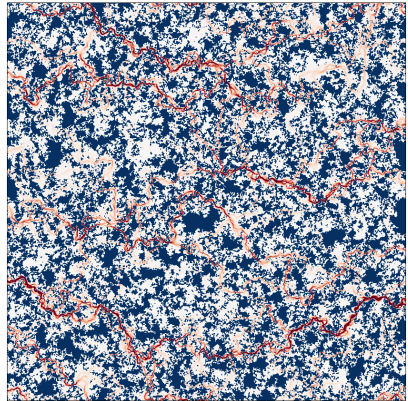
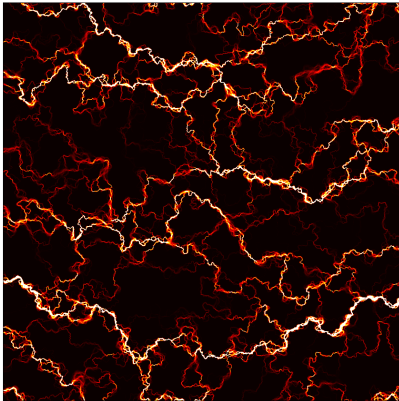
$k_t=16, k_s=32, H=0.8$
Contact area, $a(x,y)$



[1] Yastrebov, Anciaux, Molinari, Int. J. Solids Struct. 52 (2015)

● Example 2: creeping fluid transport

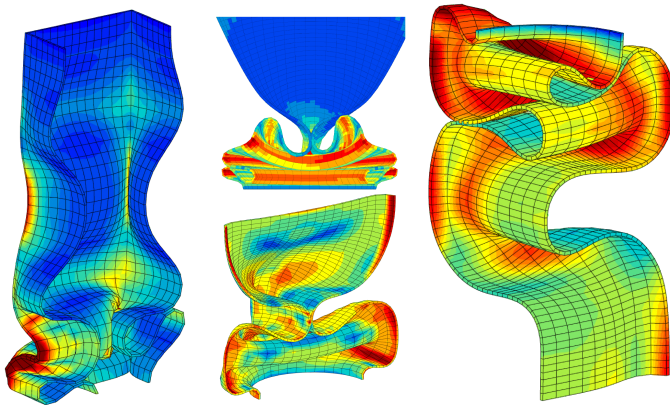
- Normal contact between rough surfaces (2048×2048 elements in BEM)
- Pressure driven creeping fluid flow simulation (FEM)
- Dynamics of flux chanel



Yastrebov, Anciaux, Molinari, Cailletaud, ICTAM 2016

• Example 3: post-buckling behavior

- Finite strain plasticity
- Buckling + self-contact

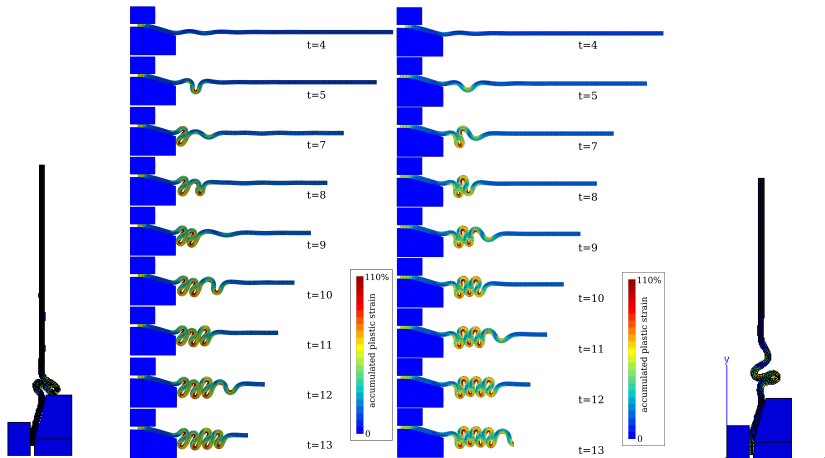


Finite-element analysis of post-buckling behavior of a thin walled structures
Z-set/ZéBuLoN

[1] Yastrebov, Numerical Methods in Contact Mechanics, Wiley/ISTE, 2013

• Example 3: post-buckling behavior

- Finite strain plasticity
- Buckling + self-contact

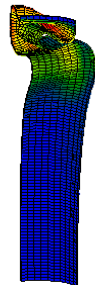
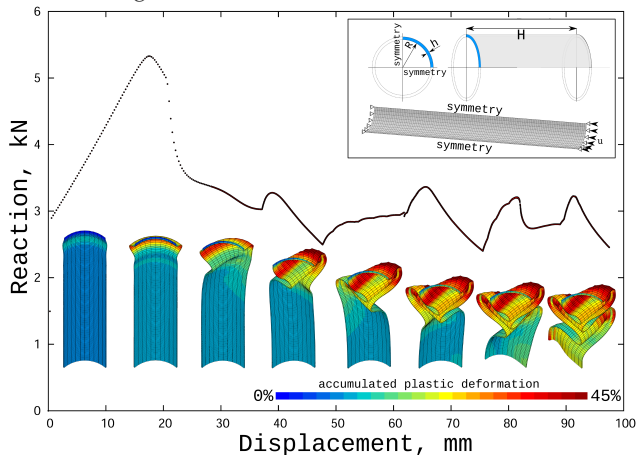


Axisymmetric finite-element analysis of post-buckling behavior of a thin walled tube (self-contact, finite strain plasticity)
Z-set/ZéBuLoN

[1] Yastrebov, Numerical Methods in Contact Mechanics, Wiley/ISTE, 2013

• Example 3: post-buckling behavior

- Finite strain plasticity
- Buckling + self-contact

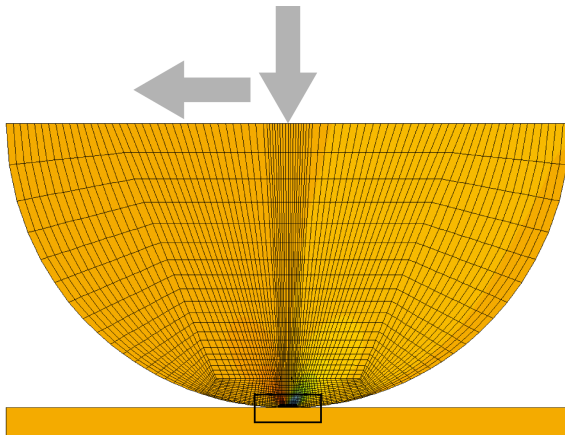


Finite-element analysis of post-buckling behavior of a thin walled tube (self-contact, finite strain plasticity)
Z-set/ZéBuLoN

[1] Yastrebov, Numerical Methods in Contact Mechanics, Wiley/ISTE, 2013

• Example 4: frictional contact

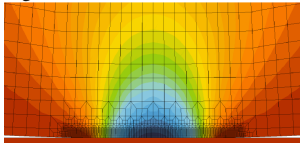
- Non-conservative problem, path dependence
- Coulomb's friction, infinitesimal deformation



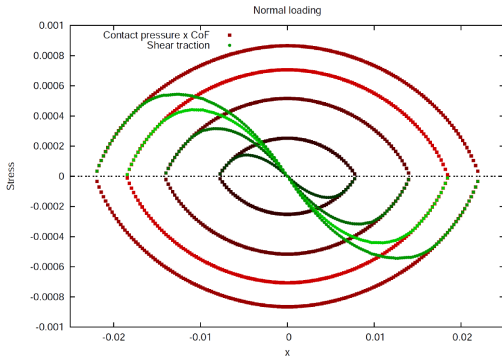
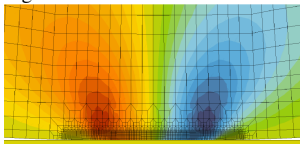
• Example 4: frictional contact

- Non-conservative problem, path dependence
- Coulomb's friction, infinitesimal deformation

sig22 at maximal normal load



sig12 at maximal normal load

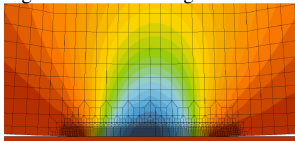


Squeezed in 100 increments, $u_z \sim t^2$

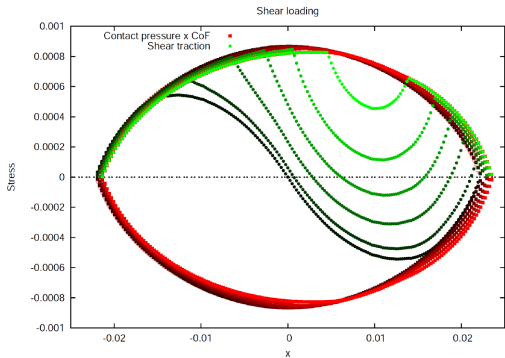
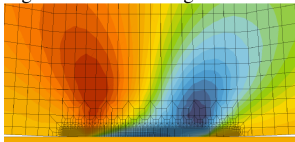
• Example 4: frictional contact

- Non-conservative problem, path dependence
- Coulomb's friction, infinitesimal deformation

sig22 at maximal tangential load



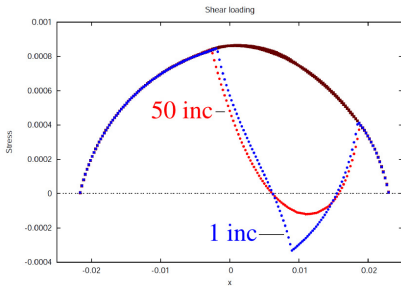
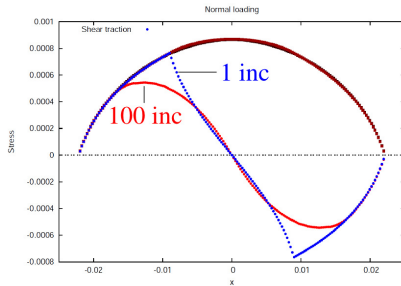
sig12 at maximal tangential load



Shifted in 100 increments, $u_x \sim t$

• Example 4: frictional contact

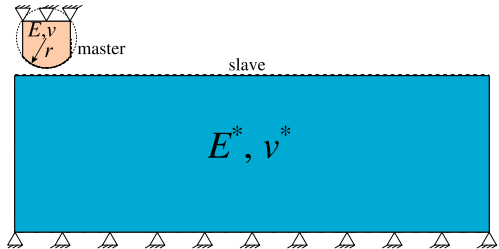
- Non-conservative problem, path dependence
- Coulomb's friction, infinitesimal deformation



Compare with a fast loading:
squeezed in 1 increment, shifted in 1 increments

• Example 5: shallow ironing test

- Frictional sliding between two deformable solids
- Problem solved by different groups^{1,2,3,4,5,6}
- Considerable dispersion of results



[1] Fischer K. A., Wriggers P., "Mortar based frictional contact formulation for higher order interpolations using the moving friction cone", Computer Methods in Applied Mechanics and Engineering, 195:5020-5036, 2006.

[2] Hartmann S., Oliver J., Cante J. C., Weyler R., Hernández J. A., "A contact domain method for large deformation frictional contact problems. Part 2: Numerical aspects", Computer Methods in Applied Mechanics and Engineering, 198:2607-2631, 2009.

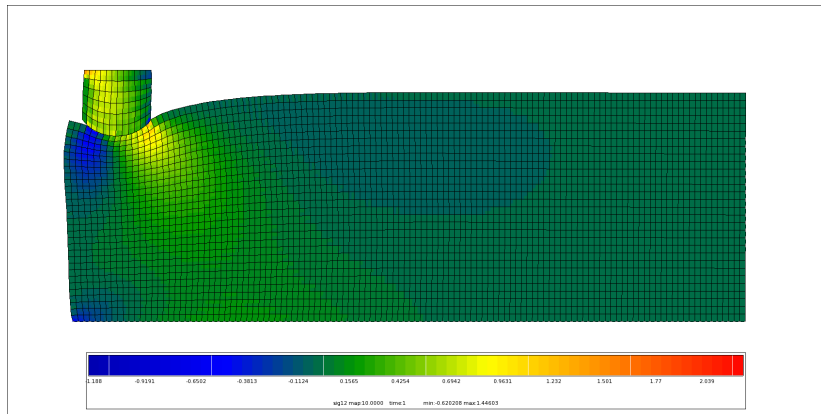
[3] Yastrebov V. A., "Computational contact mechanics: geometry, detection and numerical techniques", Thèse MINES ParisTech, 2011.

[4] Kudawoo A. D., "Problèmes industriels de grande dimension en mécanique numérique du contact : performance, fiabilité et robustesse", Thèse @ LMA & LAMSID, 2012.

[5] Poullos K., Renard Y., "A non-symmetric integral approximation of large sliding frictional contact problems of deformable bodies based on ray-tracing", Computers & Structures 153:75-90, 2015

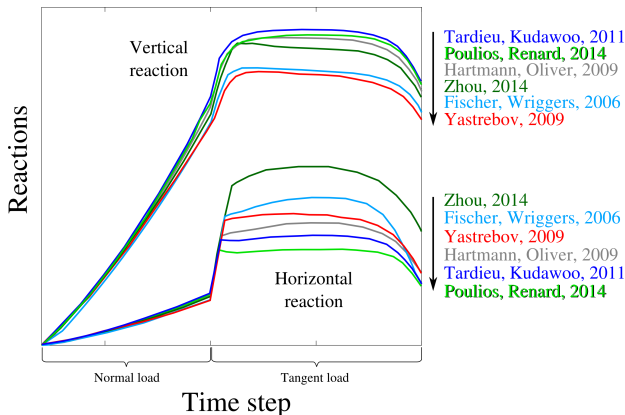
[6] Zhou Lei's blog, <http://kt2008plus.blogspot.de>

- Example 6: shallow ironing test



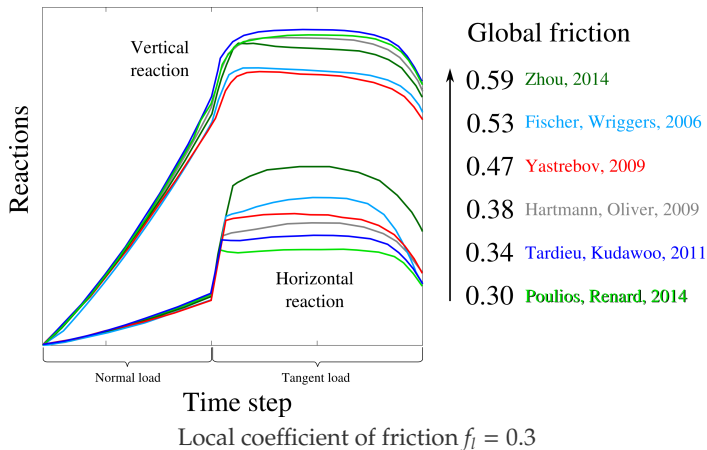
• Example 6: shallow ironing test

- No good agreement between results of different groups
- Different meshes/interpolations/large deformation formalisms



• Example 6: shallow ironing test

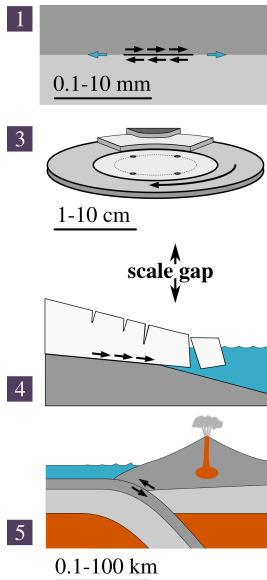
- No good agreement between results of different groups
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Elastodynamic frictional sliding on bi-material interface

Relevant applications & phenomena

- 1 Interface cracks in mixed mode
- 2 Touch interfaces
- 3 Brake systems
- scale gap*
- 4 Glacier basal slip
- 5 Slip in faults



Applications

Relevant applications & phenomena

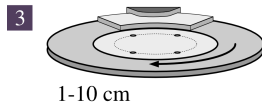
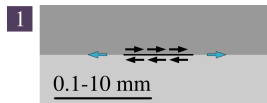
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scale gap

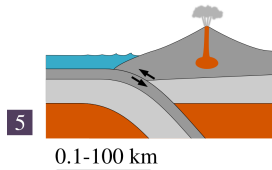
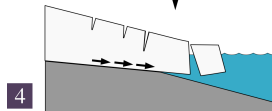
- 4 Glacier basal slip
- 5 Slip in faults

Scalable phenomena

- 1 Earthquakes can be reproduced in laboratory



scale gap



Physics of frictional sliding

- **Local** friction depends on
 - materials
 - surfaces
 - environment
 - stress state

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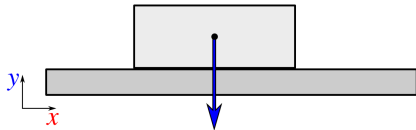
Coulomb-Amontons law

$$\begin{cases} |\tau| < f|\sigma|, & |\dot{u}| = 0 \\ |\tau| = f|\sigma|, & |\dot{u}| > 0 \end{cases}$$

τ - tangential traction,
 σ - contact pressure,
 $\dot{u}$ - slip velocity,
 f - coefficient of friction.

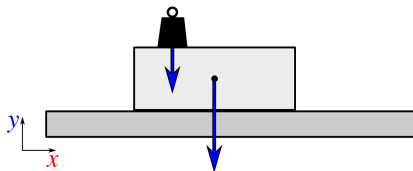
Physics of frictional sliding

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- **Global** friction depends on
 - scale
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 - geometry
- Local vs global friction



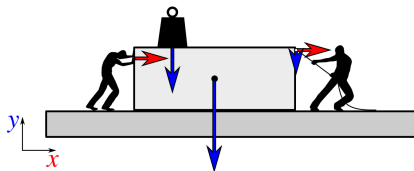
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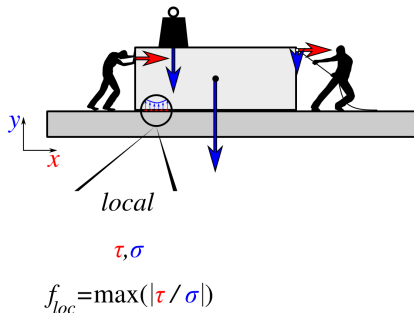
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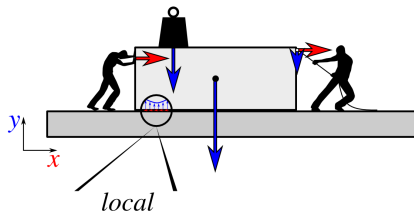
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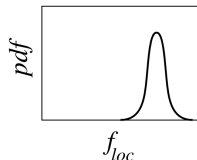
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τ, σ

$$f_{loc} = \max(|\tau / \sigma|)$$



Physics of frictional sliding

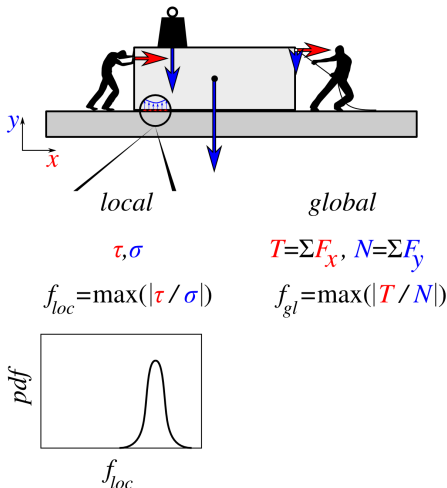
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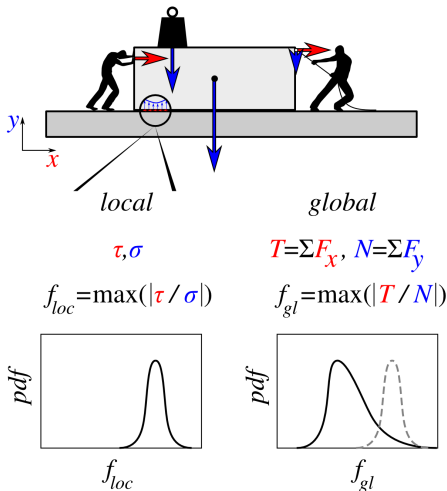
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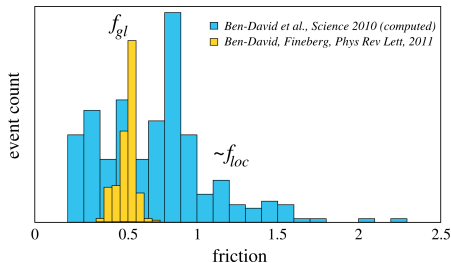
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- Local vs global friction



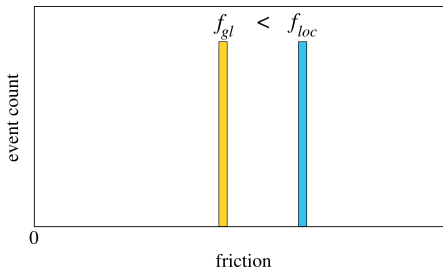
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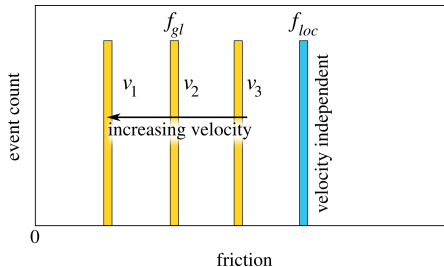


(Weertman, 1980) Unstable slippage across a fault that separates elastic media of different elastic constants, *J Geo Research*

(Adams, 1995) Self-excited oscillations of two elastic half-spaces sliding with a constant coefficient of friction, *J Appl Mech*

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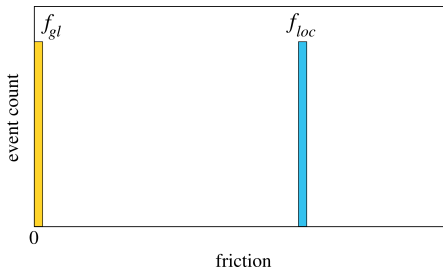


(Adams, 2000) Radiation of body waves induced by the sliding of an elastic half-space against a rigid surface, *J Appl Mech*

(Moirot, Nguyen, Oueslati, 2002) An example of stick-slip and stick-slip-separation waves, *Europ J Mech A Solids*

Physics of frictional sliding

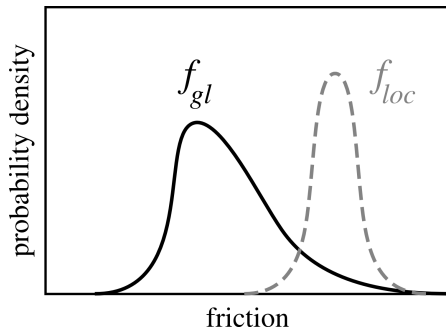
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(Cochard & Rice, 2001) Fault rupture between dissimilar materials: Ill-posedness, regularization, and slip-pulse response, J Geophys Res

Physics of frictional sliding

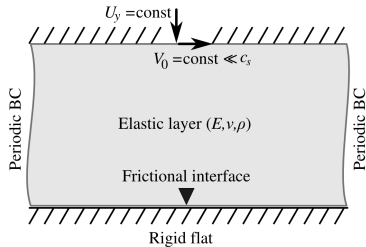
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- What are the relevant local friction laws?
- How does the dynamics affect the frictional slip?
- How can it be used for practice?

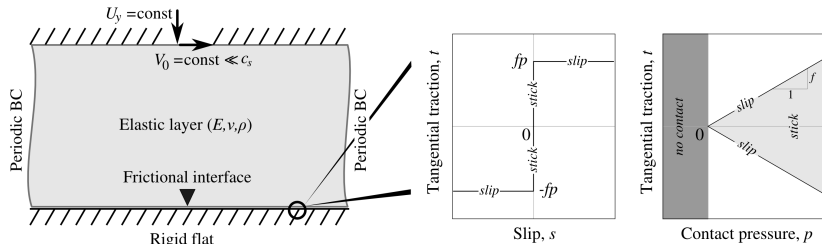
Problem set-up

- **Problem:** Elastic layer sliding on a rigid flat under Coulomb friction.



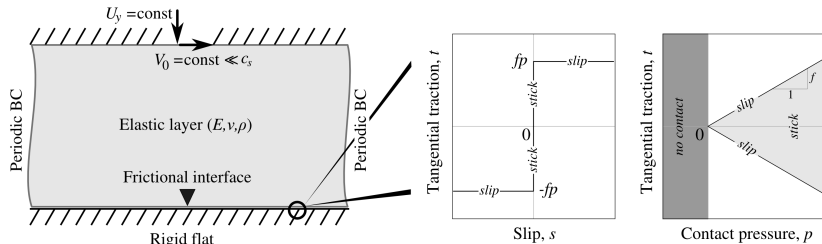
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Problem set-up

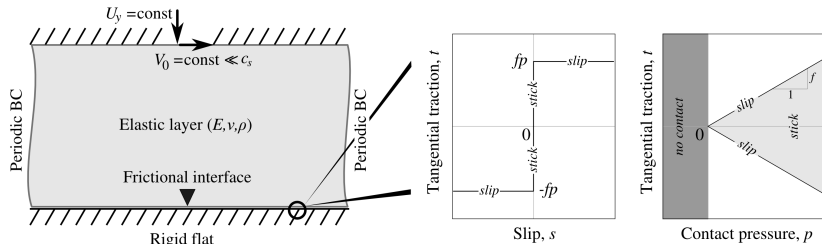
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Problem set-up

- **Problem:** Elastic layer sliding on a rigid flat under Coulomb friction.



- **Parameters:** Poisson's ratio ν , friction f and sliding velocity V_0 .
- **Methods:** Implicit dynamic finite element simulation, α -method HHT^[1], direct method to solve frictional contact^[2,3].

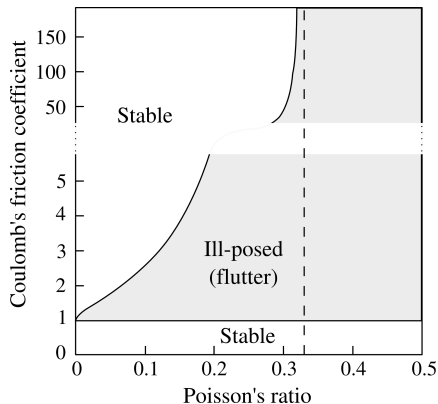
FE mesh: 33 000 quadrilateral elements with reduced integration.

[1] (Hilber, Hughes and Taylor, 1977) Improved numerical dissipation for time integration algorithms in structural dynamics, *Earthq Eng Struct D*

[2] (Francavilla & Zienkiewicz, 1975) A note on numerical computation of elastic contact problems, *Int J Num Meth Eng*.

[3] (Jean, 1995) Frictional contact in collections of rigid or deformable bodies: numerical simulation of geomaterial motions, *Stud Appl Mech*

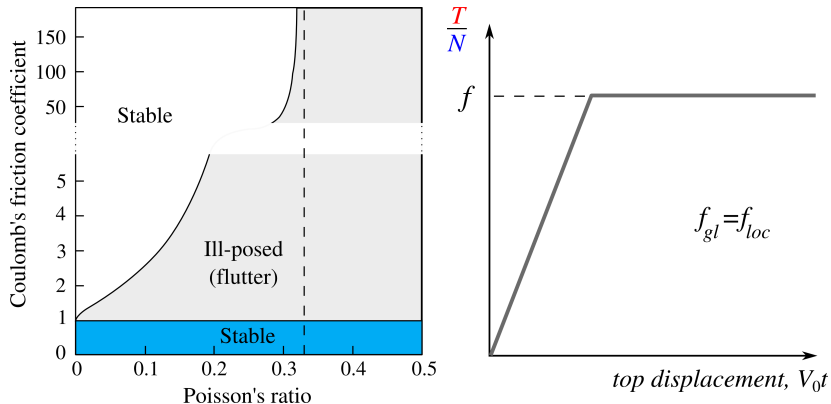
Limitations and expectations



(Renardy, 1992 [1989]) Ill-posedness at the boundary for elastic solids sliding under Coulomb friction, *J Elast*

(Martins, Guimarães & Faria, 1995 [1993]) Dynamic Surface Solutions in Linear Elasticity and Viscoelasticity With Frictional Boundary Conditions, *J Vibr Acoust*

Limitations and expectations

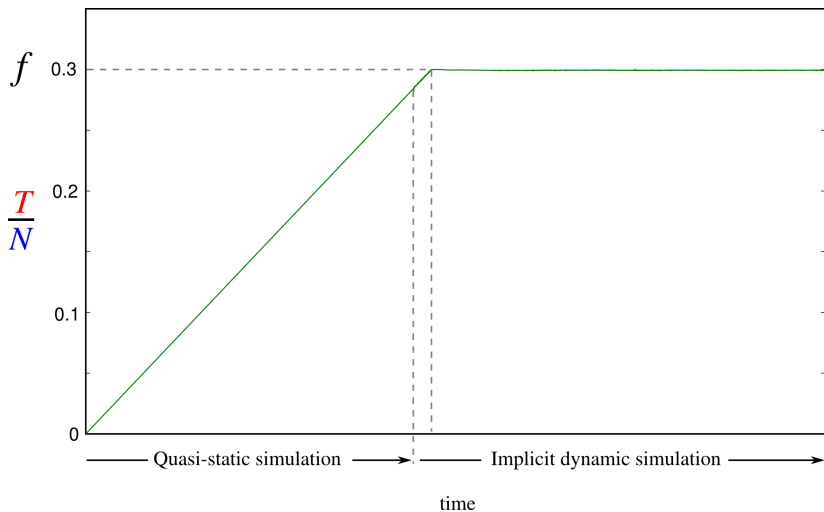


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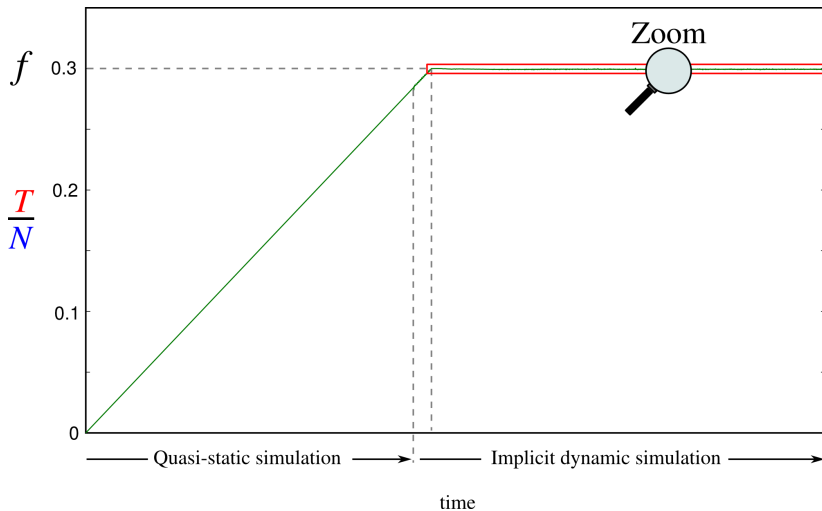
Results I: $f = 0.3$

Parameters: $\nu = 0.2, f = 0.3, V_0 = 10^{-6}c_s$.



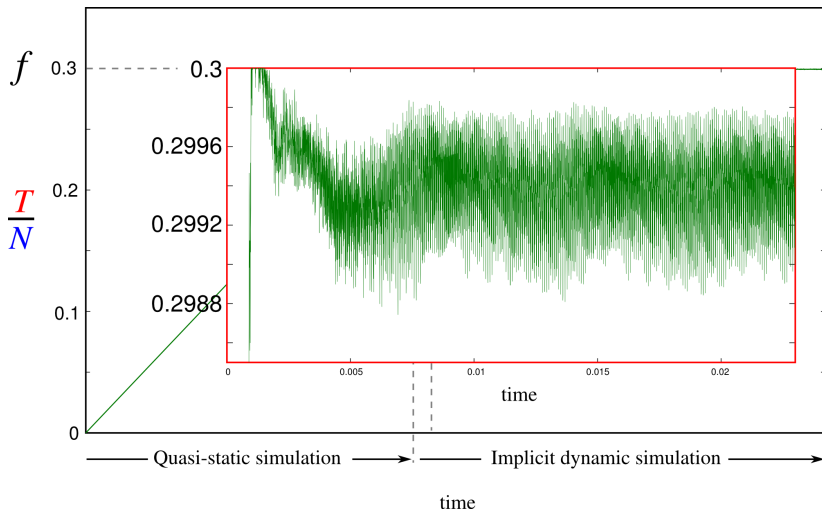
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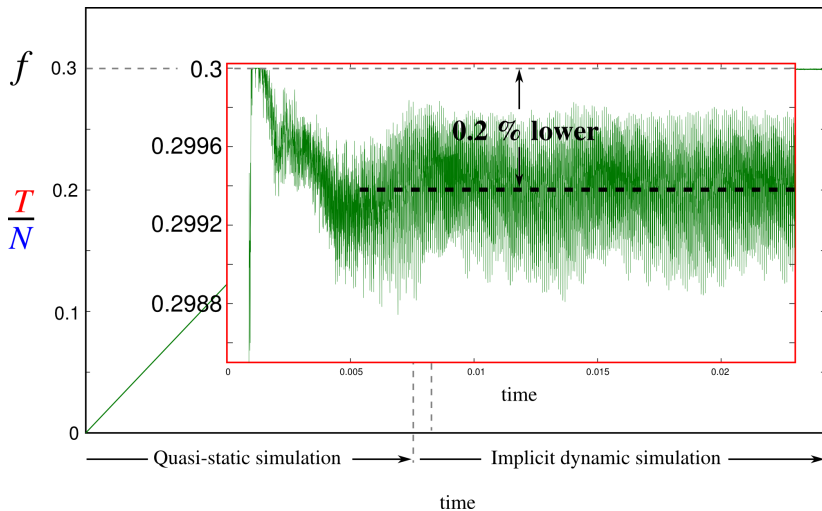
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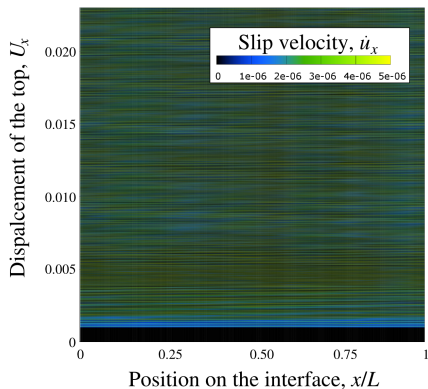
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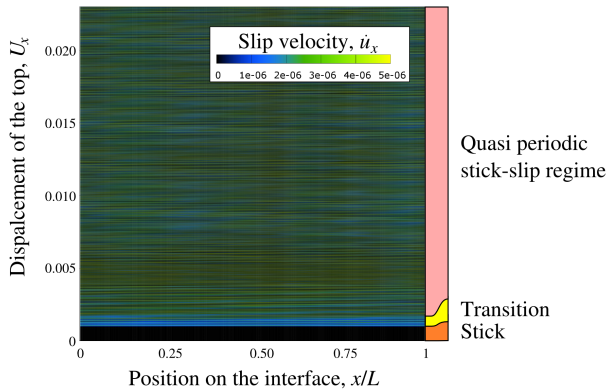
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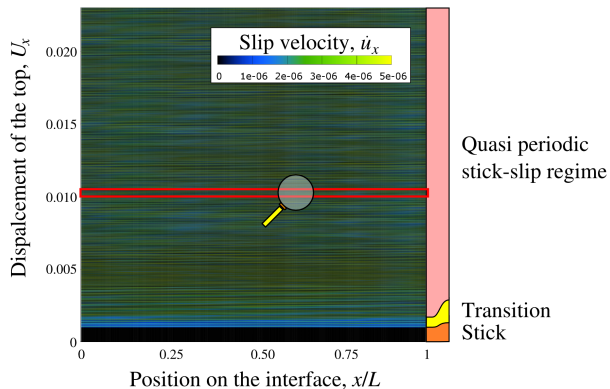
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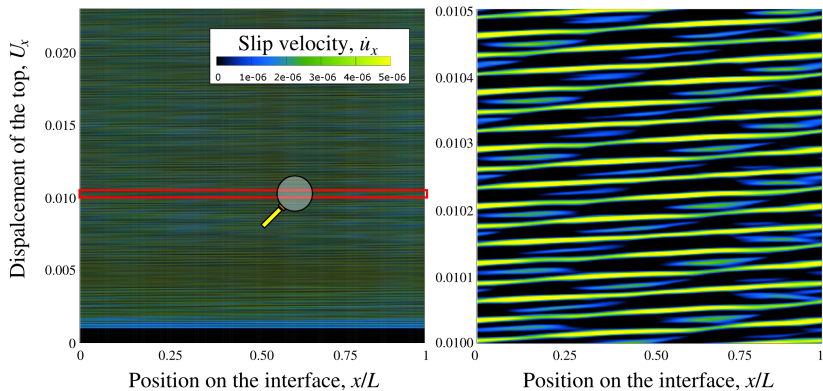
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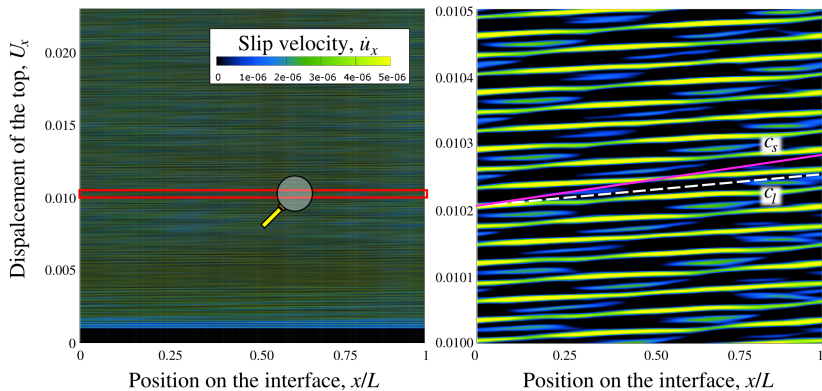
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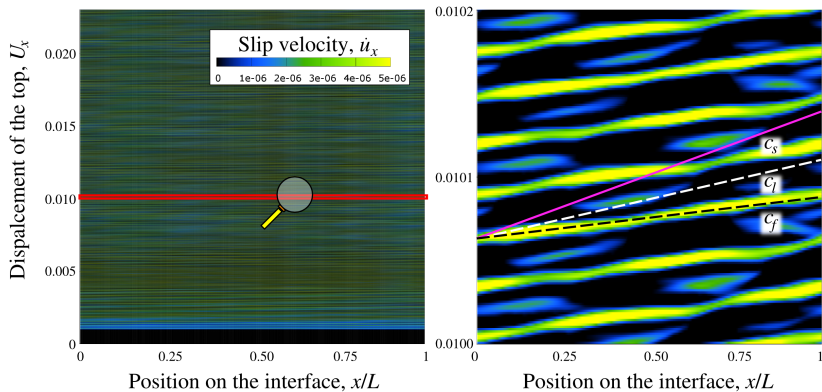
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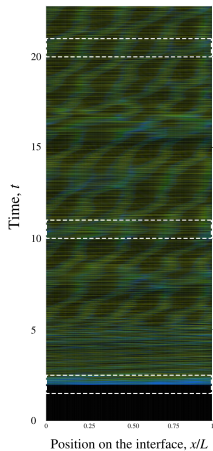
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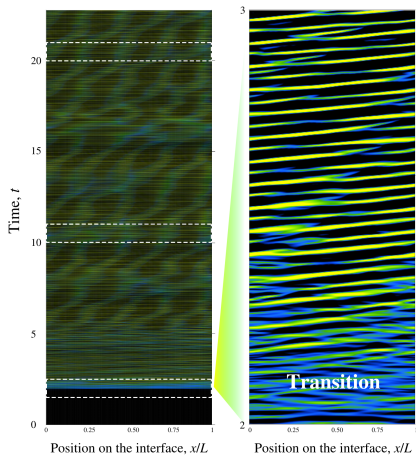
Results II: $f = 0.6$

Parameters: $\nu = 0.2, f = 0.6, V_0 = 10^{-6}c_s$.



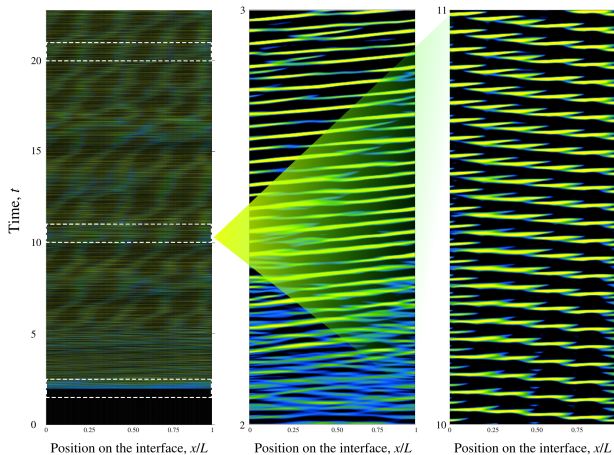
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Parameters: $\nu = 0.2, f = 0.6, V_0 = 10^{-6}c_s$.



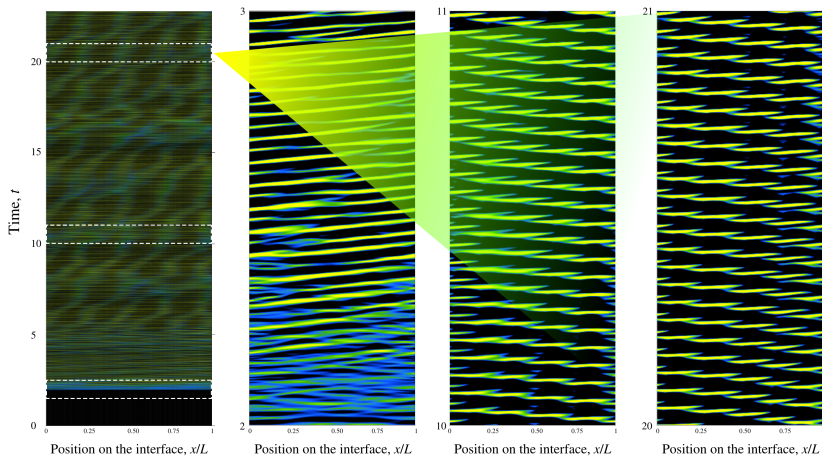
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Parameters: $\nu = 0.2, f = 0.6, V_0 = 10^{-6}c_s$.

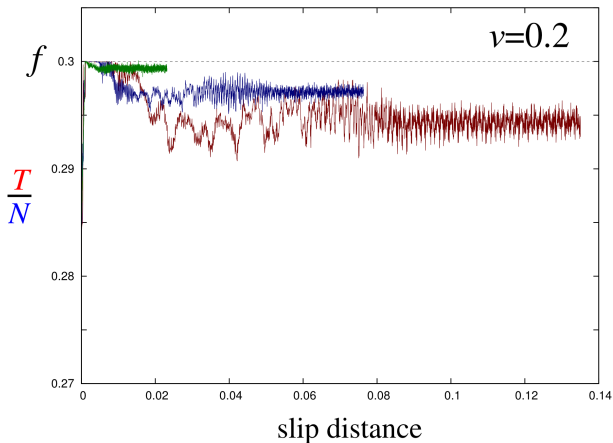
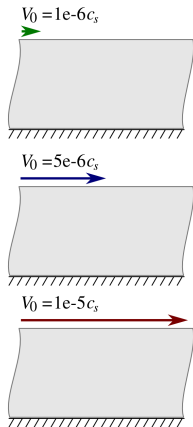


Results II: $f = 0.6$

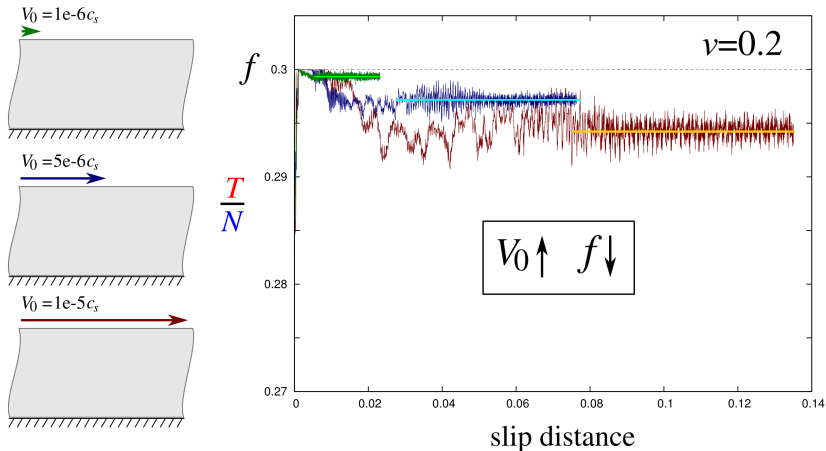
Parameters: $\nu = 0.2, f = 0.6, V_0 = 10^{-6}c_s$.



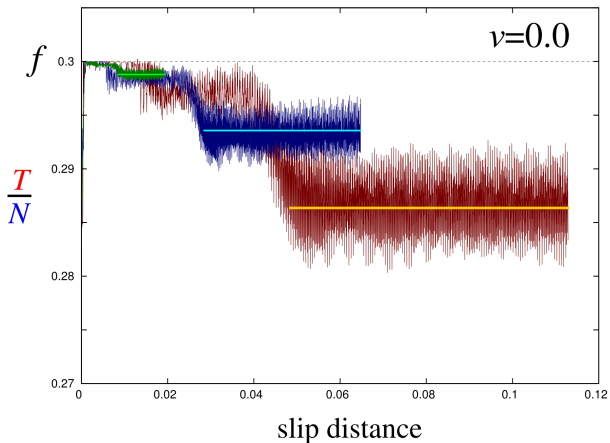
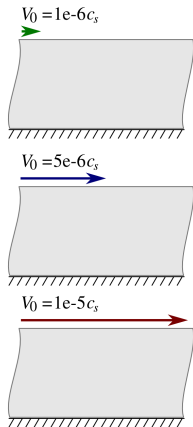
Results III: velocity dependence



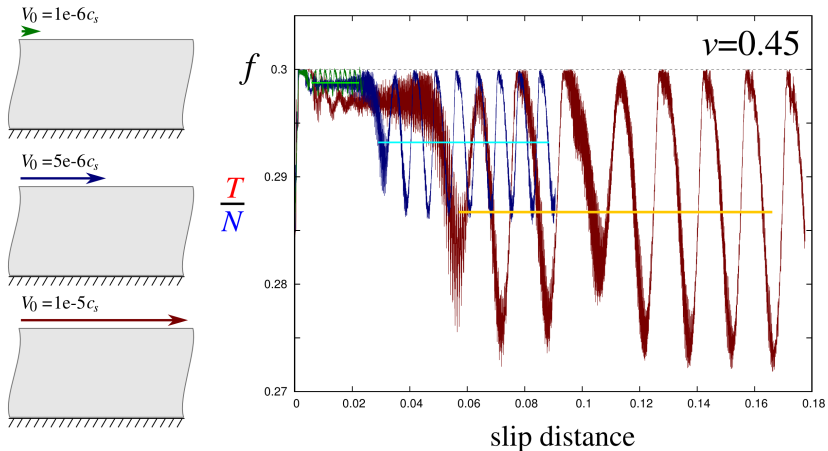
Results III: velocity dependence



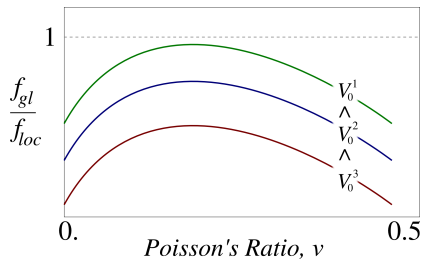
Results III: velocity dependence



Results III: velocity dependence



Results III: velocity dependence



- For a train of rectangular stick-slip waves (**Adams, 2000**) predicted velocity dependence:

$$f_{gl} = f_{loc} - \left(\frac{f}{f_{loc}} - 1 \right) \frac{S\alpha\dot{u}_x G}{(L-S)\sigma_{yy}c_s},$$

where S is the slip length, L period, G shear modulus, $f = \sigma_{xy}/\sigma_{yy}$ in body waves, $\alpha = \alpha(c_f, c_l, f)$ a coefficient.

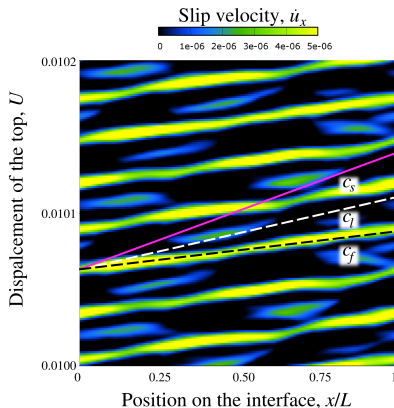
(Adams, 2000) Radiation of body waves induced by the sliding of an elastic half-space against a rigid surface, *J Appl Mech*

Results I: discussion

Intermediate discussion:

- uniform slip in *finite size systems* is unstable
- stick-slip waves
- intersonic $c_s < c_p \leq c_l$ and supersonic pulses $c_p > c_l$
- global friction is reduced

$$f_{gl} < f_{loc}$$



Results I: discussion

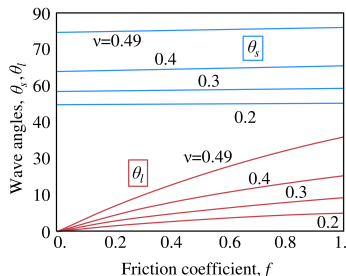
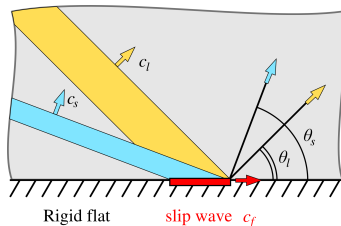
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- global friction is reduced

$$f_{gl} < f_{loc}$$

Explanation:

- Waveguide modes^[1,2]
- Stick-slip waves^[3]
- “Radiation of body waves induced by the sliding”^[4]



[1] (Mindlin, 1955) An introduction to the mathematical theory of vibrations of elastic plates.

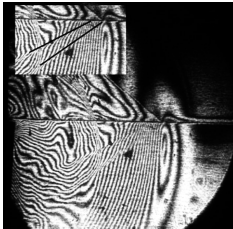
[2] (Brener et al., 2016) Dynamic instabilities of frictional sliding at a bimaterial interface, J Mech Phys Solids

[3] (Bui & Oueslati, 2010) On the stick-slip waves under unilateral contact . . . , Ann Solid Struct Mech

[4] (Adams, 2000) Radiation of body waves induced by the sliding of an elastic half-space . . . , J Appl Mech

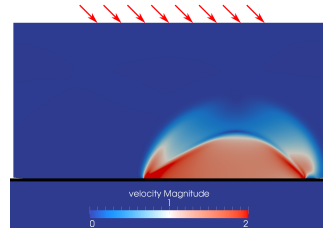
Results I: discussion II

Supersonic slip propagation does not violate causality
even though c_l is the maximal signal speed



(Coker, Rosakis, Needleman, 2003) Dynamic crack growth along a polymer composite-Homalite interface, *J Mech Phys Solids*

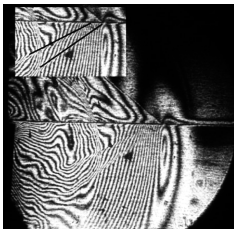
(Coker, Lykotrafitis, Needleman, Rosakis, 2005) Frictional sliding modes along an interface between identical elastic plates subject to shear impact loading, *J Mech Phys Solids*



(Kammer & Yastrebov, 2012) On the Propagation of Slip Fronts at Frictional Interfaces, *Tribol Lett*

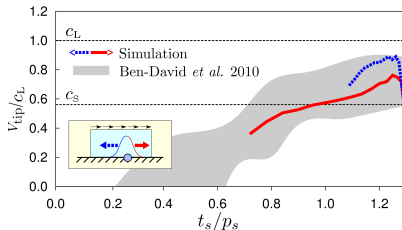
Results I: discussion II

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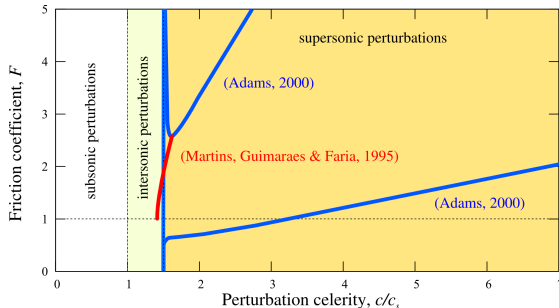


(Kammer & Yastrebov, 2012) On the Propagation of Slip Fronts at Frictional Interfaces, *Tribol Lett*

(Ben-David, Cohen & Fineberg 2010) The dynamics of the onset of frictional slip, *Science*

Results I: discussion III

Inter- and supersonic stable and unstable roots for $\nu = 0.1$



(Martins, Guimarães & Faria, 1995) Dynamic Surface Solutions in Linear Elasticity . . . , *J Vib Acoust*
(Adams, 2000) Radiation of body waves induced by the sliding of an elastic half-space . . . , *J Appl Mech*

■ Neglect supersonic perturbations in half-space

(Ranjith & Rice, 2001) Slip dynamics at an interface between dissimilar materials, *J Mech Phys Solids*

(Bui & Oueslati, 2010) On the stick-slip waves under unilateral contact and Coulomb friction, *Ann Solid Struct Mech*

■ Neglect supersonic perturbations in an elastic-layer

(Brener et al., 2016) Dynamic instabilities of frictional sliding at a bimaterial interface, *J Mech Phys Solids*

Within the ill-posed region

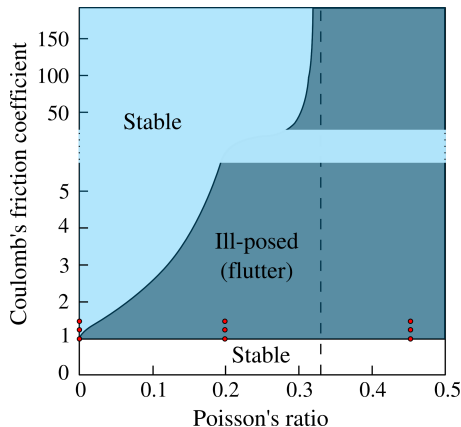
Preliminary comments:

- Ill-posed^[1] if

$$\sigma_{xy} = f\sigma_{yy}$$

holds all along the interface

- Exponential growth of amplitude results in a stick-slip or separation
- Critical region $f \geq 1$ for high frequencies
 $kH \gg 1$ ^[2-4]



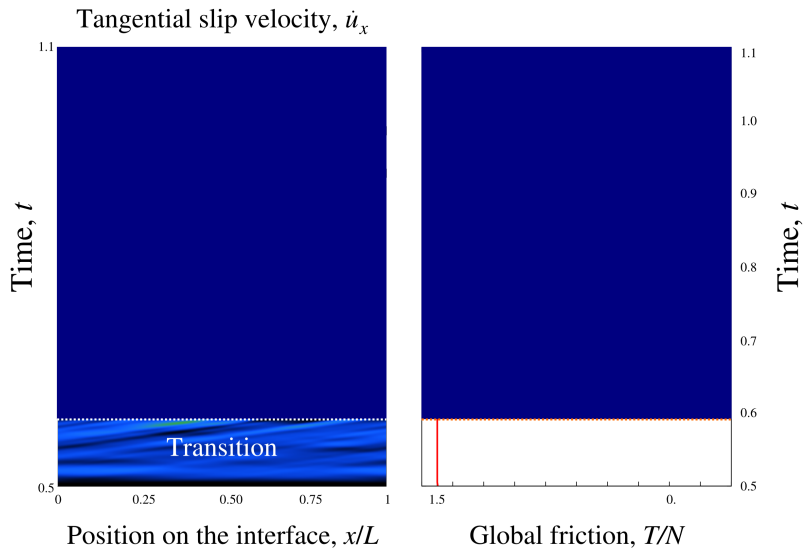
[1] (Martins, Guimarães & Faria, 1995) Dynamic Surface Solutions in Linear Elasticity . . . , *J Vibr Acoust*

[2] (Ranjith & Rice, 2001) Slip dynamics at an interface between dissimilar materials, *J Mech Phys Solids* 49

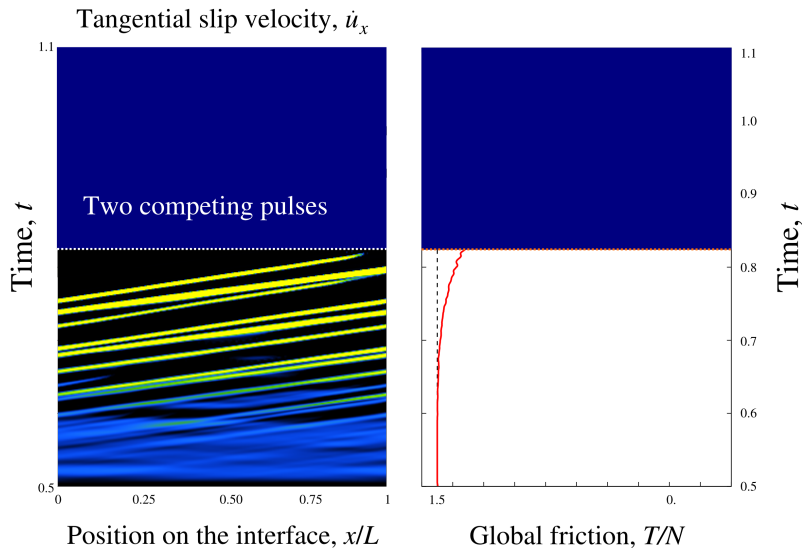
[3] (Cochard & Rice, 2000) Fault rupture between dissimilar materials: Ill-posedness . . . , *J Geophys Res* 105

[4] (Kammer, Yastrebov, Anciaux, and Molinari, 2014) The existence of a critical length scale in regularised friction, *J Mech Phys Solids* 63

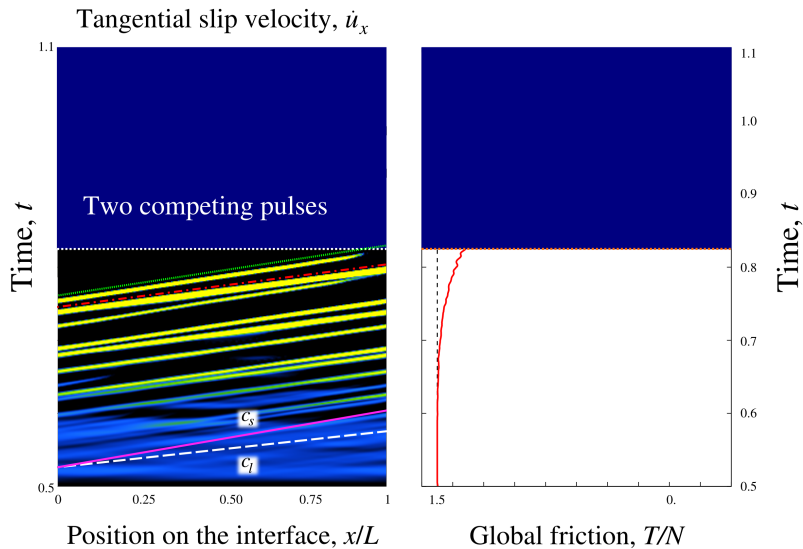
Results IV: within the ill-posed region $f = 1.5$



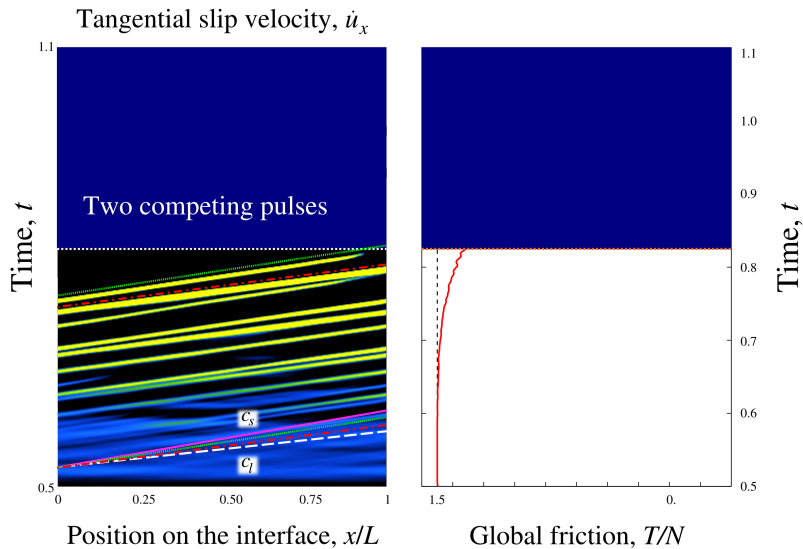
Results IV: within the ill-posed region $f = 1.5$



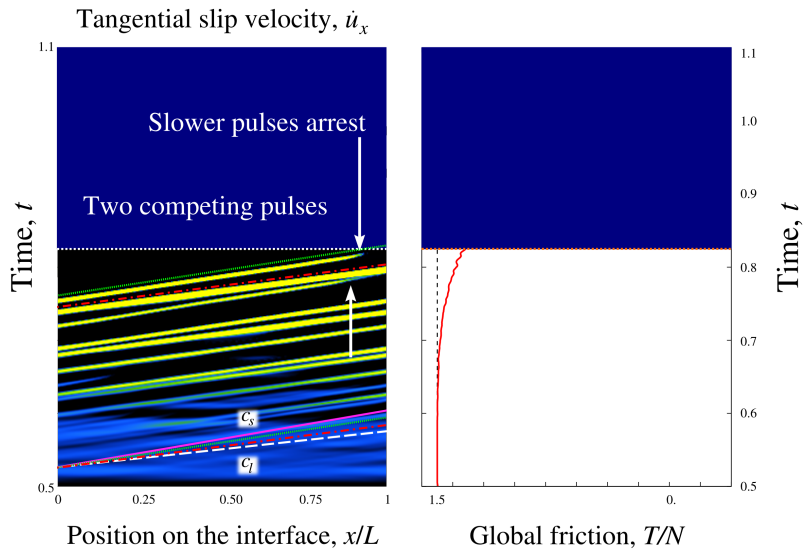
Results IV: within the ill-posed region $f = 1.5$



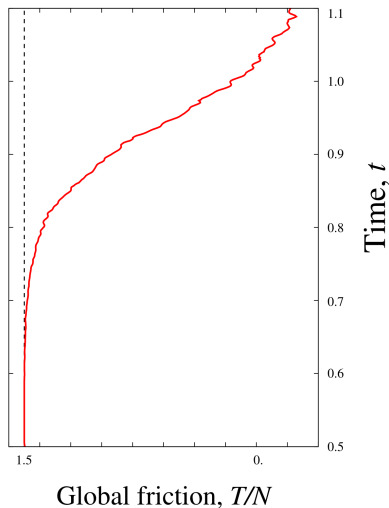
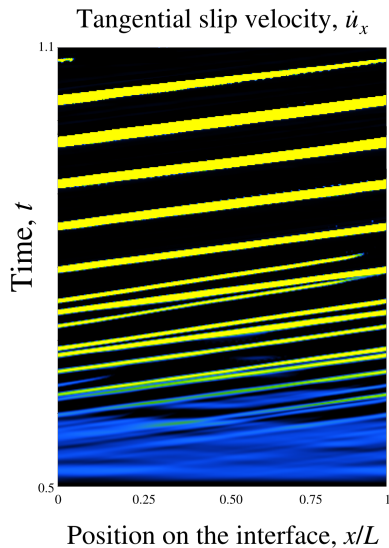
Results IV: within the ill-posed region $f = 1.5$



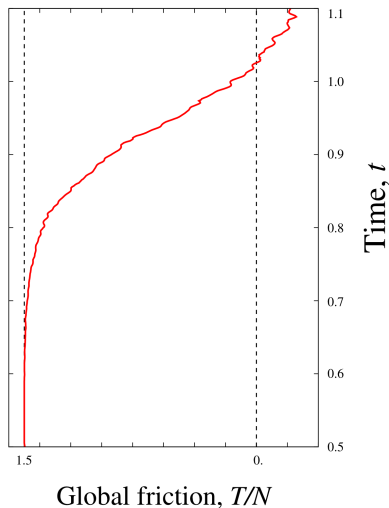
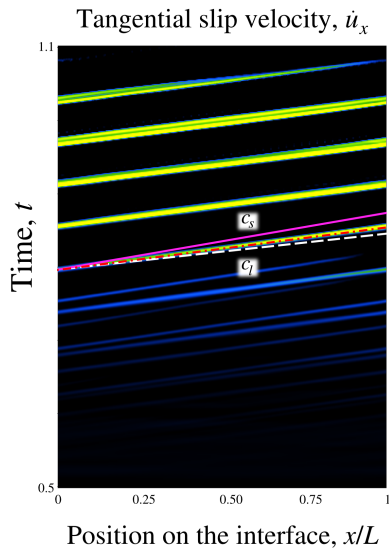
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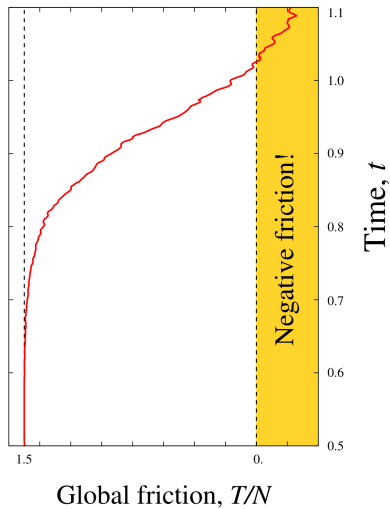
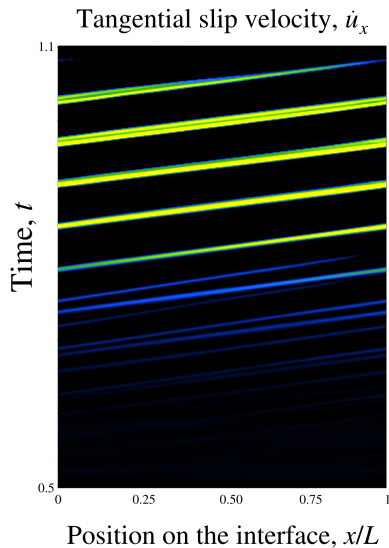
Results IV: within the ill-posed region $f = 1.5$



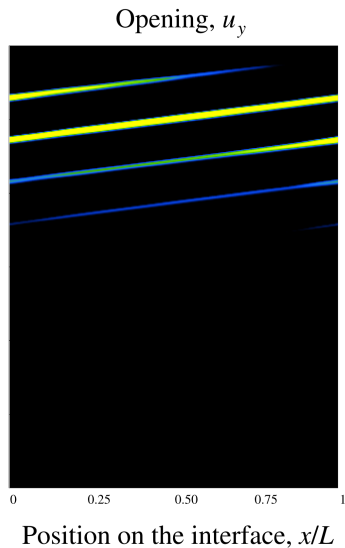
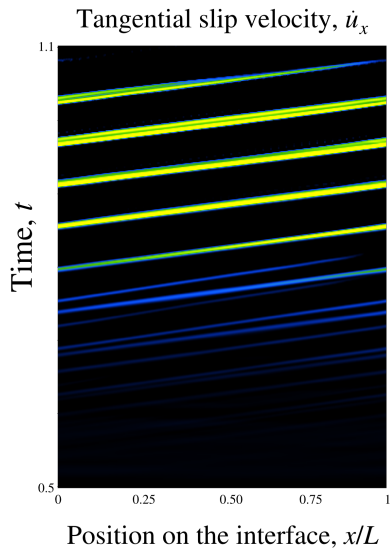
Results IV: within the ill-posed region $f = 1.5$



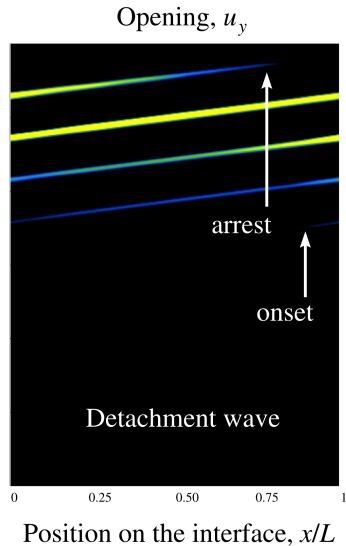
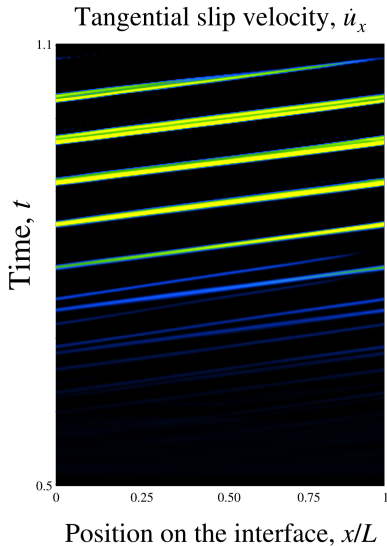
Results IV: within the ill-posed region $f = 1.5$



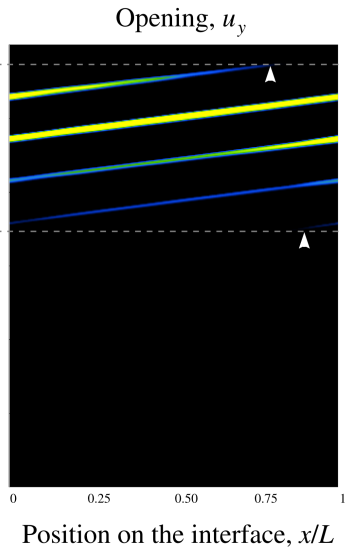
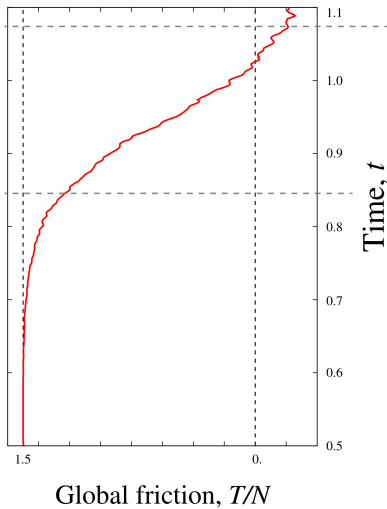
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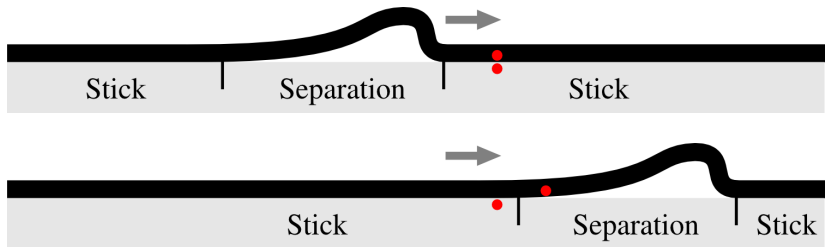
Results IV: within the ill-posed region $f = 1.5$



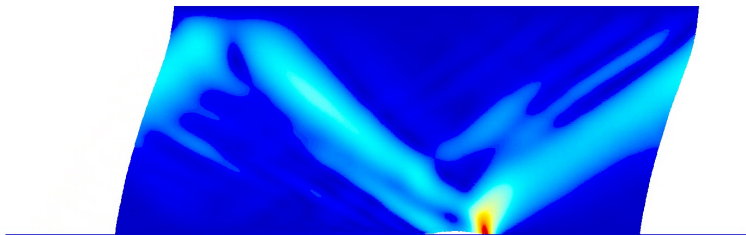
Results IV: within the ill-posed region $f = 1.5$



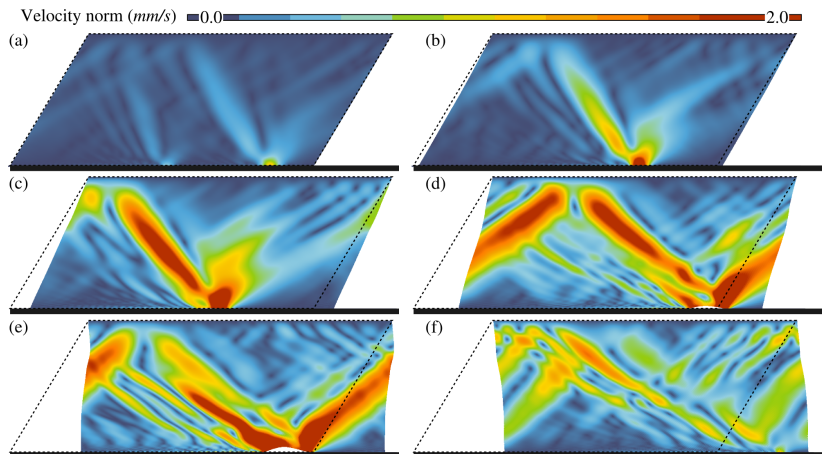
Results IV: animation



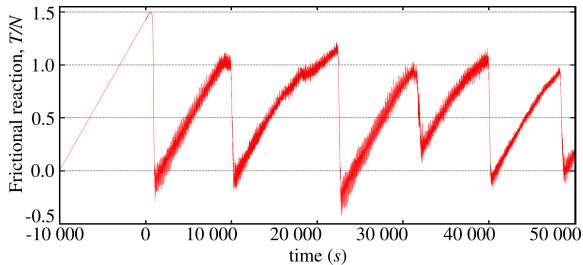
Results IV: animation



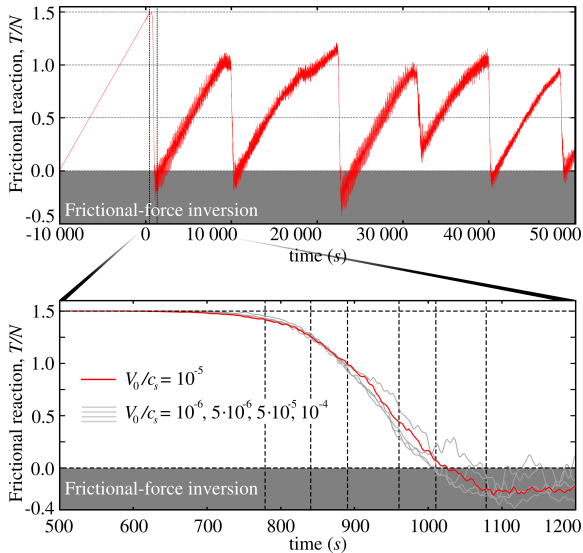
Results IV: animation



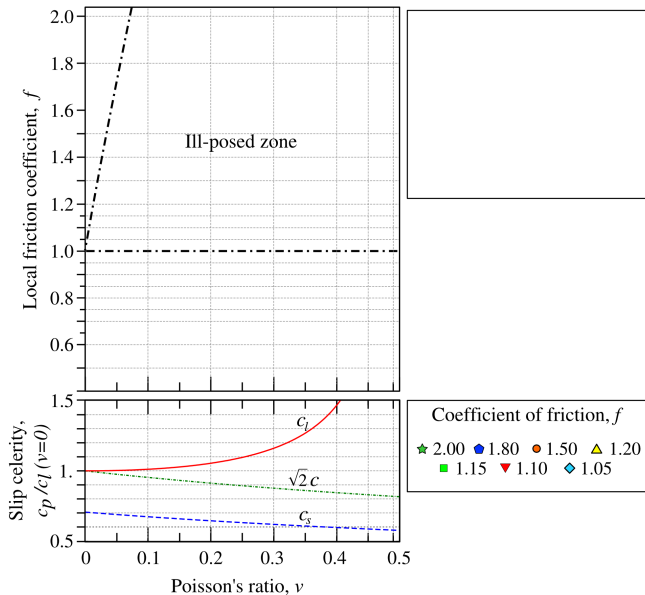
Results IV: behavior at longer periods



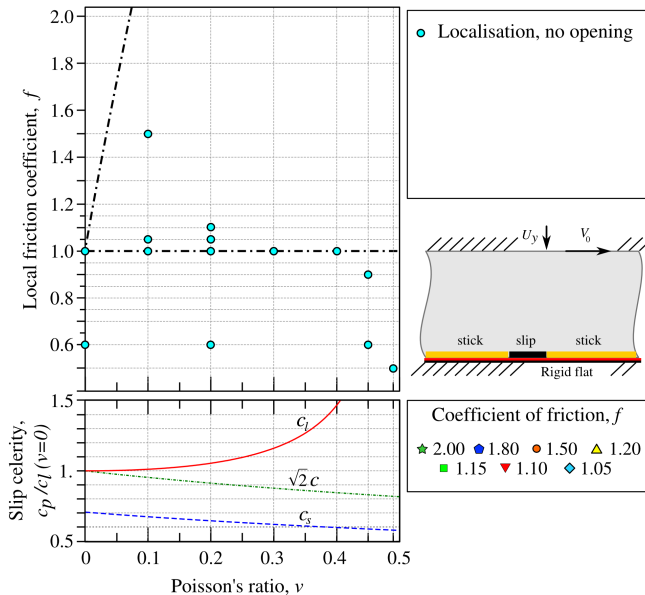
Results IV: behavior at longer periods



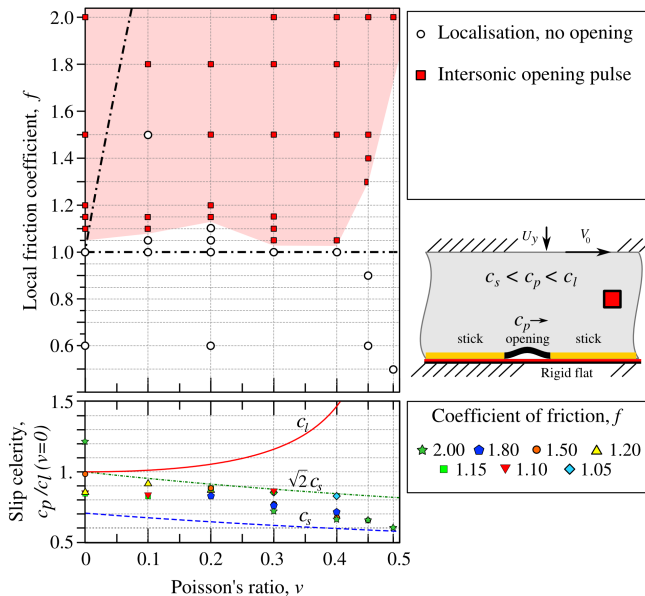
Results IV: parametric study



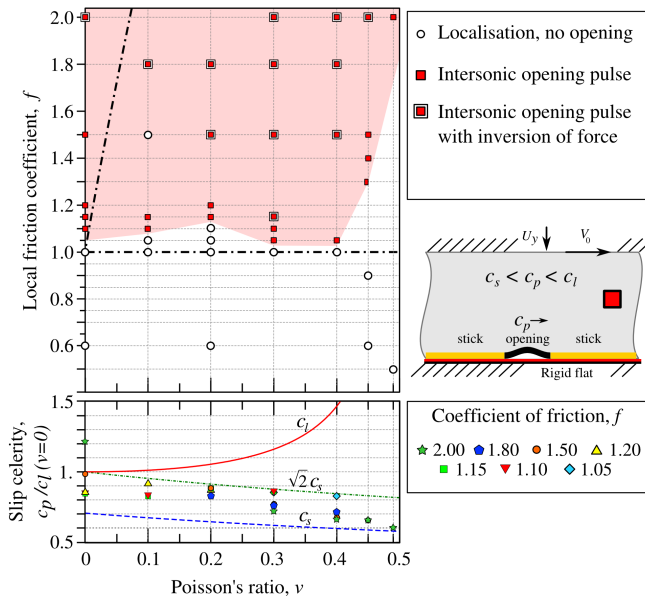
Results IV: parametric study



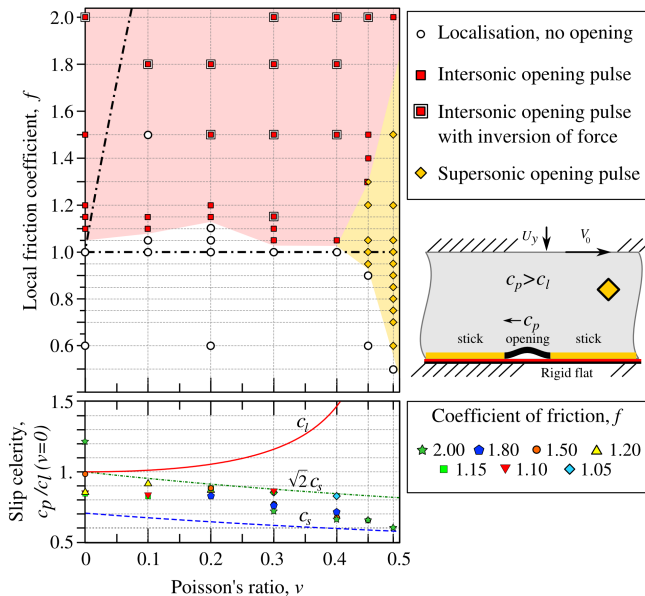
Results IV: parametric study



Results IV: parametric study



Results IV: parametric study



Formal analysis I

- Scalar elastodynamic potentials for dilatational and shear waves^[1]:

$$\varphi = f(y) \exp[ik(x - ct)], \quad \psi = g(y) \exp[ik(x - ct)]$$

$$f(y) = A \sin(\eta_d y) + B \cos(\eta_d y), \quad g(y) = C \sin(\eta_s y) + D \cos(\eta_s y)$$

- Horizontal and vertical displacement:

$$u = [ikf(y) - g'(y)] \exp[ik(x - cy)], \quad v = [f'(y) + ikg(y)] \exp[ik(x - cy)]$$

- Stress components:

$$\sigma_{xx} = -\mu \left[(k^2 \gamma^2 + \xi^2 \eta_d^2) f + 2ikg' \right] \exp[ik(x - cy)]$$

$$\sigma_{yy} = -\mu \left[(\xi^2 k^2 + \gamma^2 \eta_d^2) f - 2ikg' \right] \exp[ik(x - cy)]$$

$$\sigma_{xy} = \mu \left[\xi^2 (\eta_s^2 - k^2) g + 2ikf' \right] \exp[ik(x - cy)]$$

$$\sigma_{zz} = \nu(\sigma_{xx} + \sigma_{yy}) = -\mu \xi^2 f (k^2 + \eta_d^2) \exp[ik(x - cy)]$$

- Constants:

$$\gamma^2 = c_d^2 / c_s^2, \quad \xi^2 = \lambda / \mu$$

λ, μ are Lamé parameters, c_s, c_l are transverse and longitudinal wave celerities.

[1] (Miklowitz, 1980) The Theory of elastic waves and waveguides. North-Holland

- Prescribed displacements on boundary $y = H$:

$$\begin{cases} u(H) = ikf(H) - g'(H) = 0, \\ v(H) = f'(H) + ikg(H) = 0, \end{cases} \quad (1)$$

- Zero vertical displacement on boundary $y = 0$ (no opening):

$$v(H) = f'(0) + ikg(0) = 0$$

- Frictional relation between normal and shear tractions:

$$\sigma_{xy}(0) + F\sigma_{yy}(0) = \xi^2(\eta_s^2 - k^2)g(0) + 2ikf'(0) - F[(\xi^2k^2 + \gamma^2\eta_d^2)f(0) - 2ikg'(0)] = 0$$

- Obtain a linear system of equations for $X = \{A, B, C, D\}$

$$KX = 0$$

- For nontrivial solutions, we require that

$$\det(K) = 0$$

Formal analysis III

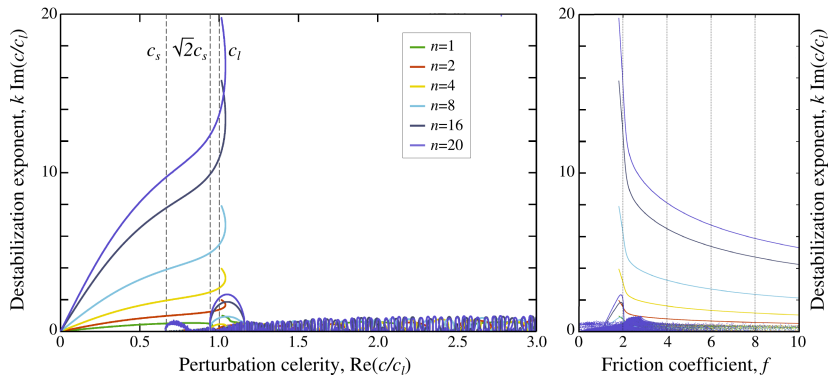
- $\det(K) = 0$ corresponds to this transcendental equation:

$$\begin{aligned} & F \left[\sin(\eta_d H) + \frac{\eta_s \eta_d}{k^2} \sin(\eta_s H) \right] \left(-2 \frac{\eta_d \eta_s}{k^2} \sin(\eta_d H) + (\xi^2 + \gamma^2 \frac{\eta_d^2}{k^2}) \sin(\eta_s H) \right) - \\ & - F \frac{\eta_s \eta_d}{k^2} [\cos(\eta_d H) - \cos(\eta_s H)] \left(2 \cos(\eta_d H) + (\xi^2 + \gamma^2 \frac{\eta_d^2}{k^2}) \cos(\eta_s H) \right) + \\ & + i \frac{\eta_d}{k} \left[2 + \xi^2 \left(\frac{\eta_s^2}{k^2} - 1 \right) \right] \left(\cos(\eta_d H) \sin(\eta_s H) + \frac{\eta_s \eta_d}{k^2} \cos(\eta_s H) \sin(\eta_d H) \right) = 0 \end{aligned}$$

- Fix ν and $k = 2\pi n/L$ with $n \in \mathbb{Z}$
- Take $\eta_d = |k| \sqrt{(c/c_d)^2 - 1}$, $\eta_s = |k| \sqrt{(c/c_s)^2 - 1}$
- Express $F = F(\text{Re}(c), \text{Im}(c))$
- Search for $\text{Re}(F)$ at lines with $\text{Im}(F) = 0$
- Solution $u \sim \exp[ik(x - \text{Re}(c)t)] \cdot \exp[k\text{Im}(c)t]$

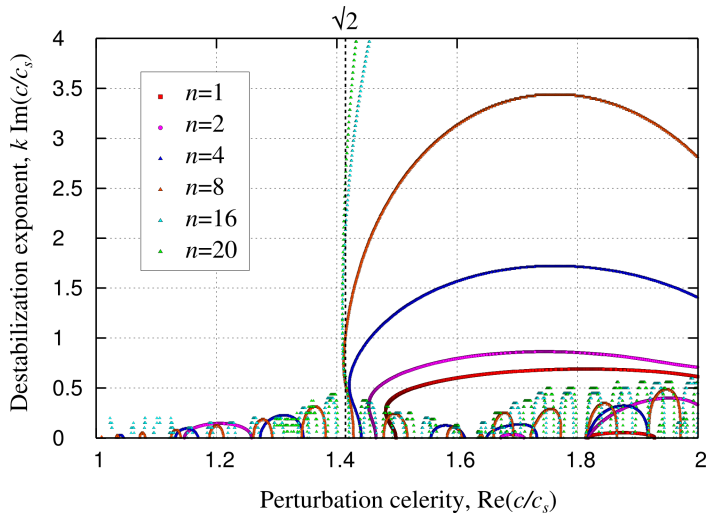
Formal analysis III

■ Example: $\nu = 0.1, L = 2H, k = 2\pi n/L$



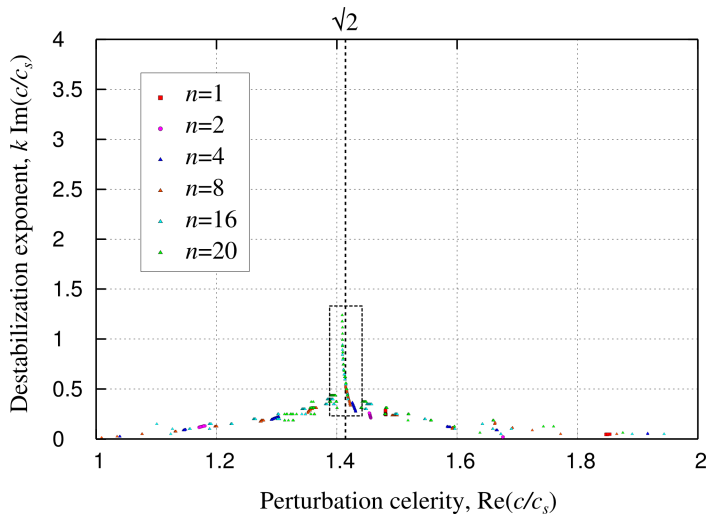
Formal analysis III

- Example: $\nu = 0.2, L = 2H, k = 2\pi n/L$



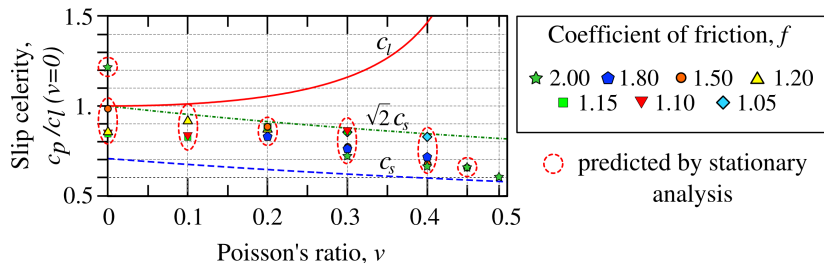
Formal analysis III

- Example: $\nu = 0.2, L = 2H, k = 2\pi n/L$

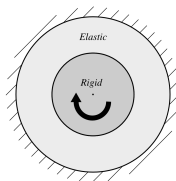
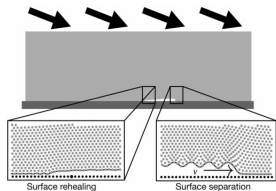


Formal analysis III

■ Example: $\nu = 0.2, L = 2H, k = 2\pi n/L$

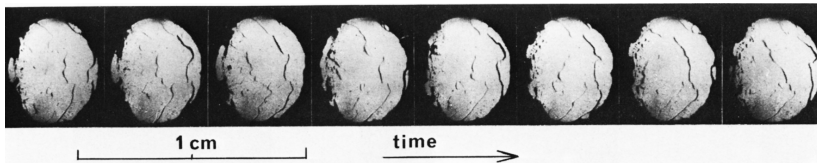


Results IV: references



(Gerde & Marder, 2001) Friction and Fracture, *Nature*

(Moirot, Nguyen, Oueslati, 2002) An example of stick-slip and stick-slip-separation waves, *Eur J Mech A-Solid*



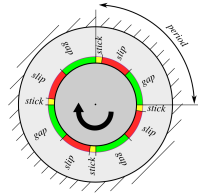
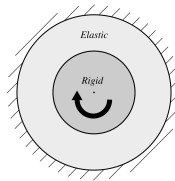
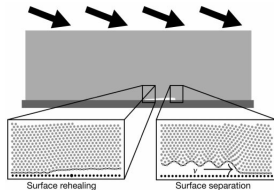
(Schallamach, 1971) How does rubber slide?, *Wear*

Remark: Schallamach waves are much slower.

(Comninou & Dundurs, 1977, 1978) Elastic interface waves involving separation

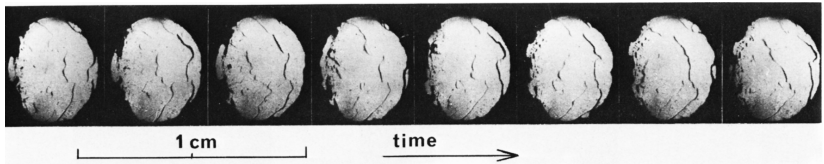
(Freund, 1978) Discussion: elastic interface waves involving separation

Results IV: references



(Gerde & Marder, 2001) Friction and Fracture, *Nature*

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(Freund, 1978) Discussion: elastic interface waves involving separation

■ Sub-critical regime

- stationary solution
- supersonic stick-slip
- or standing waves stick-slip
- velocity dependent global friction

■ Critical regime

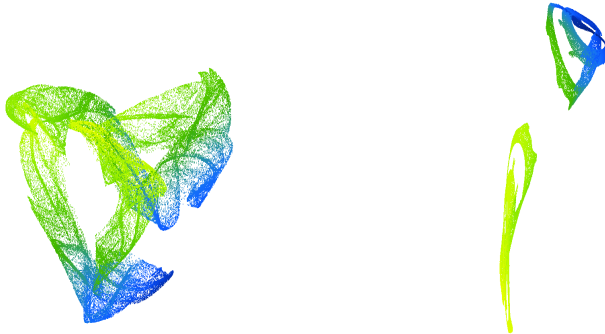
- intersonic transient slip pulse
- transforms into opening pulse
- ★ sliding without slipping(!)
- ★ inversion of frictional force(!)
- stationary waveguide analysis predicts instability and slip velocity

[1] V.A. Yastrebov, Sliding Without Slipping Under Coulomb Friction: Opening Waves and Inversion of Frictional Force, *Tribology Letters* 62:1-8 (2016)

[2] V.A. Yastrebov, Elastodynamic frictional sliding of an elastic layer on a rigid flat, *in preparation*

Vibration of asymmetric materials with internal contacts

Examples of 4D strange attractor projected on 3D $(x_1, \dot{x}_1, x_2, \dot{x}_2)$



Merci de votre attention!

[<www.yastrebov.fr>](http://www.yastrebov.fr)

P.S. Looking for a motivated postdoc ☺